

A Study on Integration of Energy Harvesting System and Semi-Active Control for a Hydraulic Suspension System

整合液壓減振器能源擷取系統及 半主動減振控制之研究

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Outline

- Introduction
- Test Rig Layout
- System Modelling and Simulation
- Controller Design
- Results and Discussions
- Conclusions



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- **Introduction**
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- **System Modelling and Simulation**
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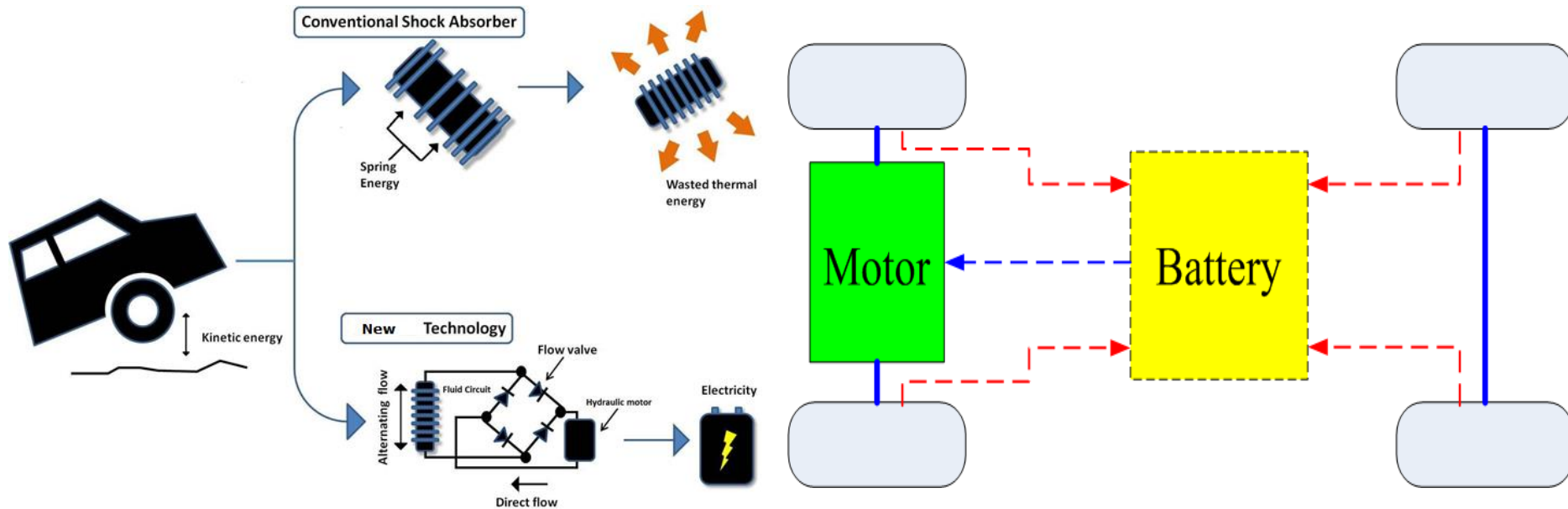
Motivation(1/4)

- Exhaustion of energy
- Green concept
- Renewable energy technologies



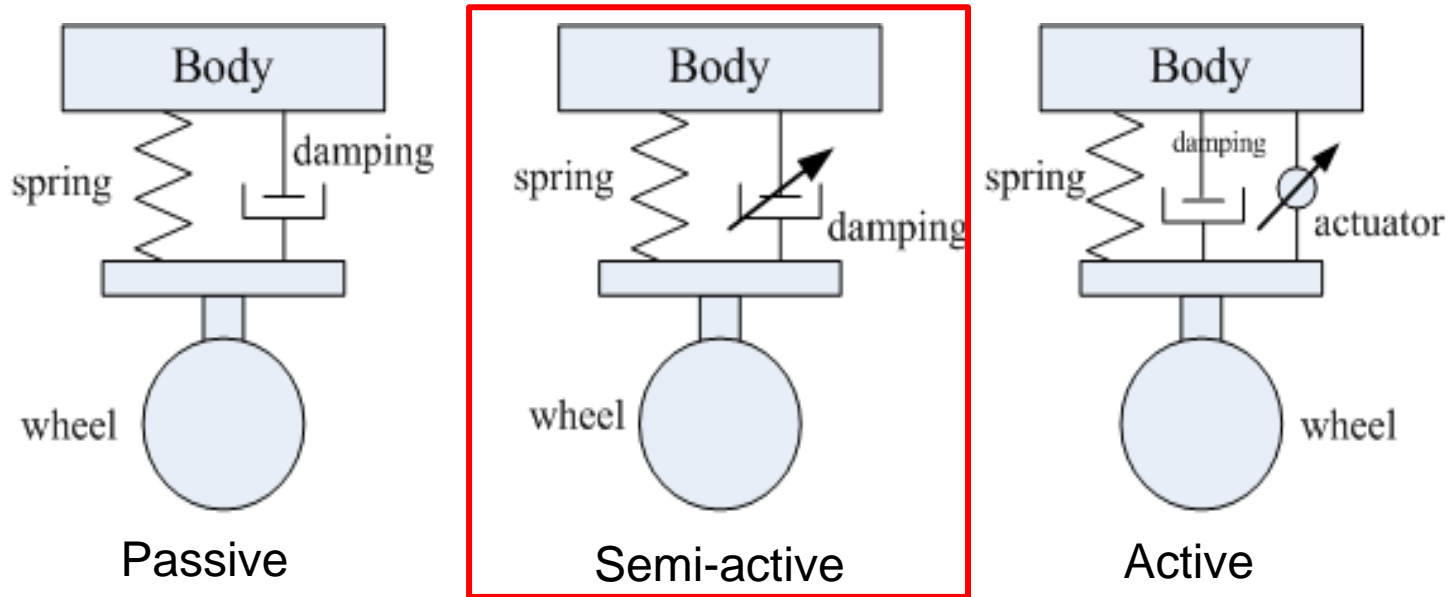
Motivation(2/4)

- Energy harvesting system of electric vehicles
- Energy efficiency



Motivation(3/4)

■ Semi-active control



Passive

- 1.Fixed damping coefficient
- 2.Simple

Semi-active

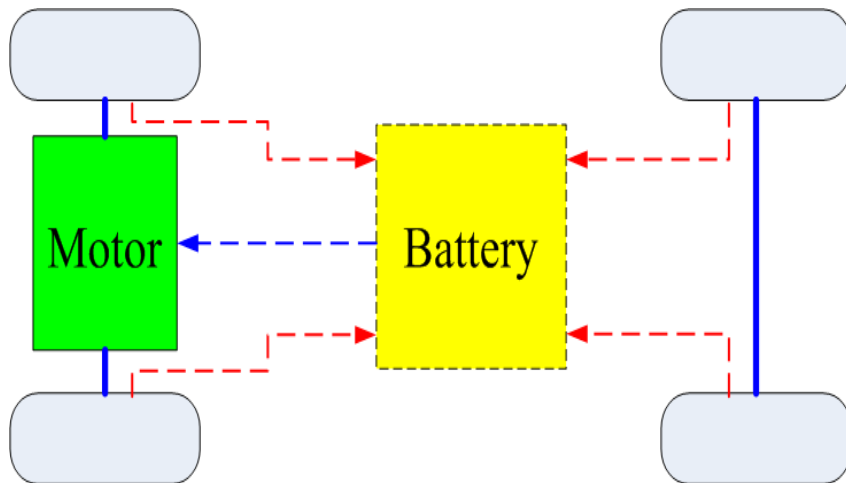
- 1.Adjustable damping coefficient
- 2.Combining advantages

Active

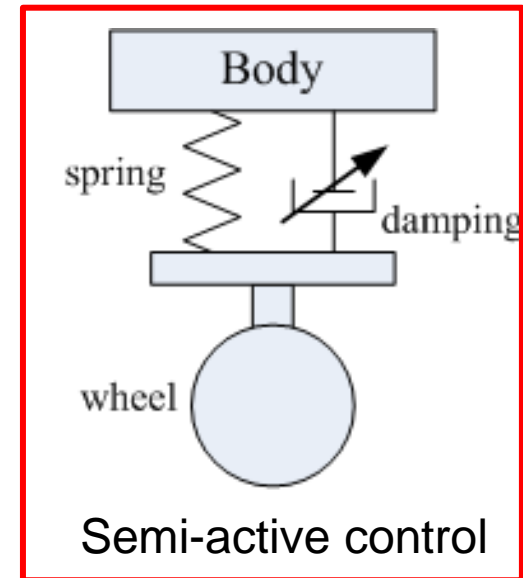
- 1.Force apply
- 2.Need external energy
- 3.Expensive

Motivation(4/4)

Energy harvesting system



Semi-active suspension



■ Semi-active suspension

In 1981, Segel and Lang [1] used the relationship of F_d-V to test the performance of damper.

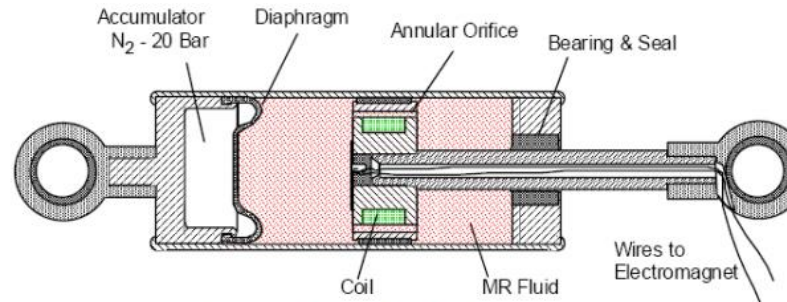


Figure.6 Schematic of MR Damper

In 2002, Yao et al.[5] proposed a **semi-active control** of vehicle suspension system with a MR damper.

Outline

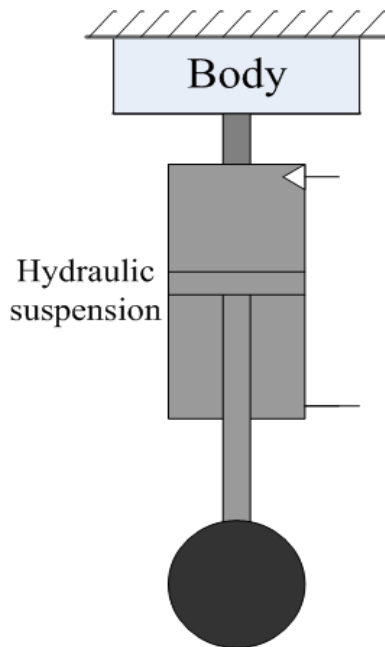
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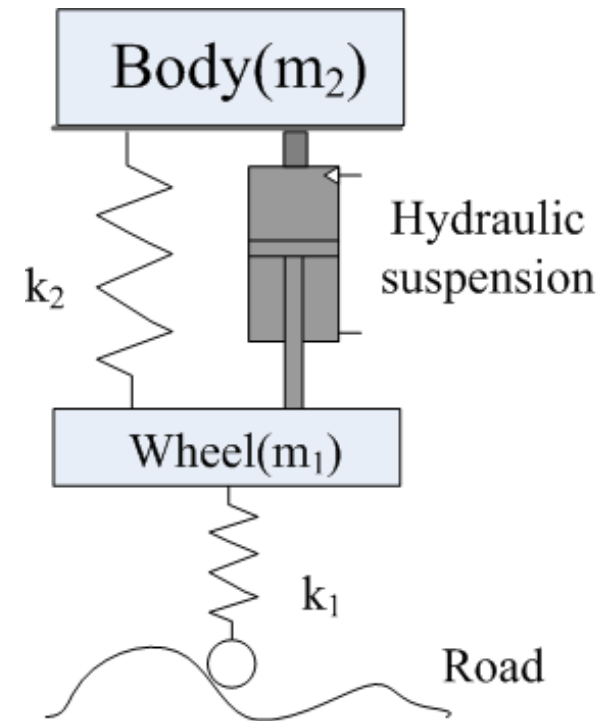
Test Rig Layout

■ Skyhook damper system

1. Throttle valve control
2. Resistance control



■ Quarter-car system in resistance control

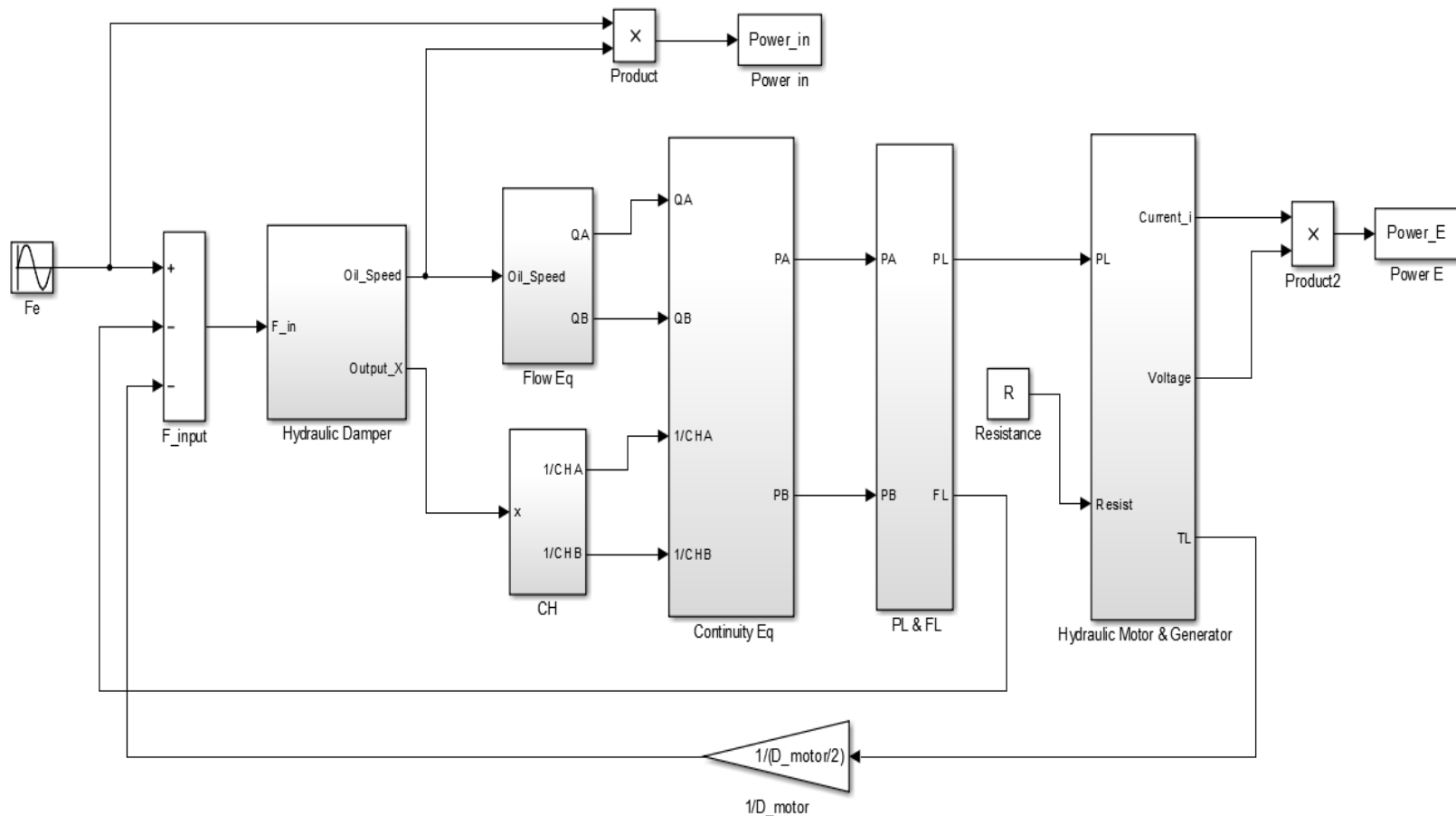


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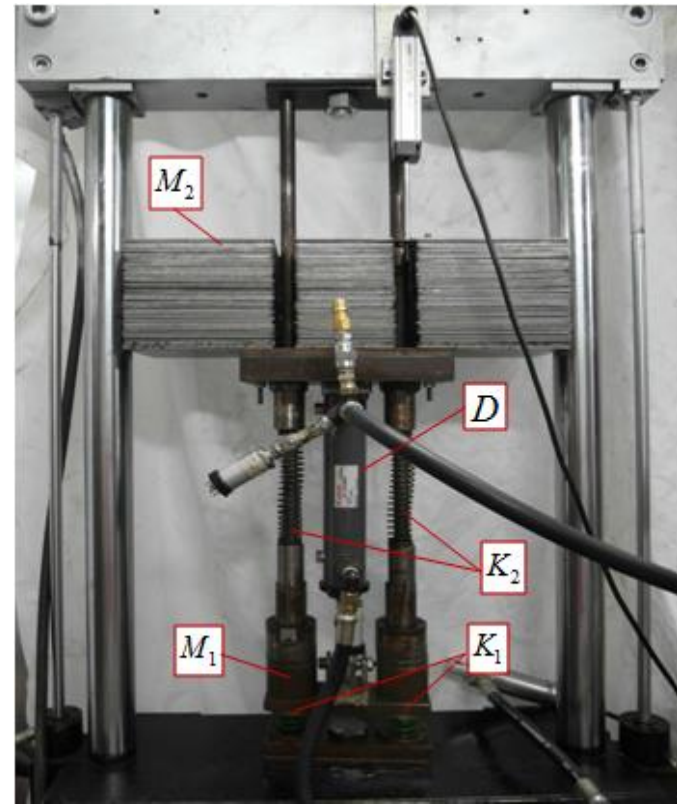
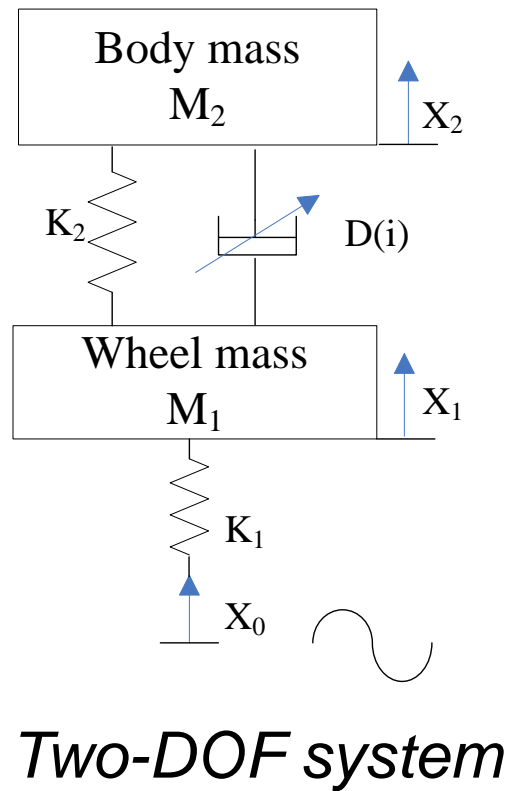


■ Skyhook damper system(4/4)



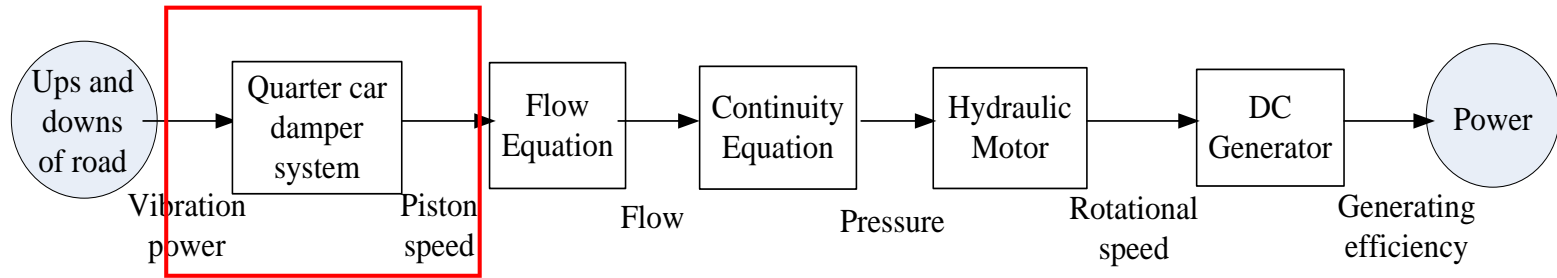
System Modelling and Simulation

■ Quarter-car system(1/3)



System Modelling and Simulation

■ Quarter-car system(2/3)

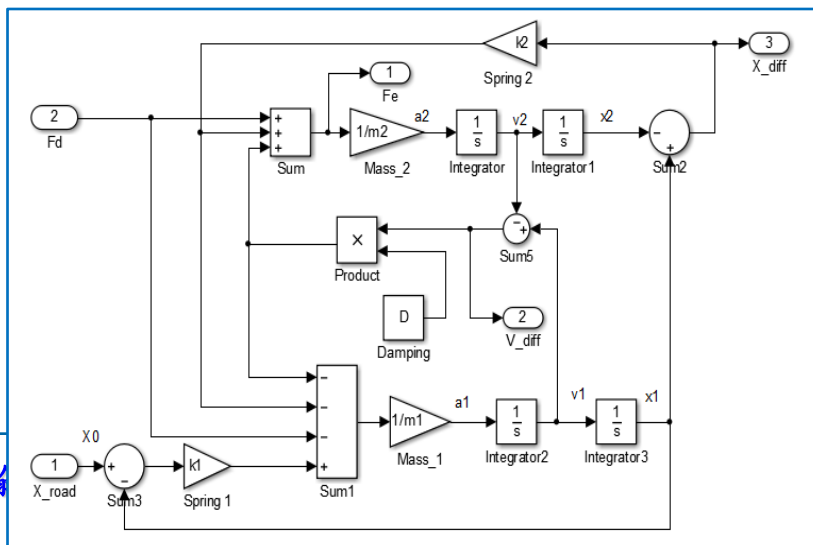


■ Dynamic equation of hydraulic damper

$$M_2 \ddot{x}_2 + D(\dot{x}_2 - \dot{x}_1) + K_2(x_2 - x_1) - F_d = 0$$

$$M_1 \ddot{x}_1 - D(\dot{x}_2 - \dot{x}_1) - K_2(x_2 - x_1) + K_1(x_1 - x_0) + F_d = 0$$

- X_1 : Wheel displacement
- X_2 : Body displacement
- M_1 : Mass of wheel
- M_2 : Mass of vehicle body
- D : Damping coefficient
- K_1 : Wheel stiffness
- K_2 : Suspension stiffness
- F_d : Damping force

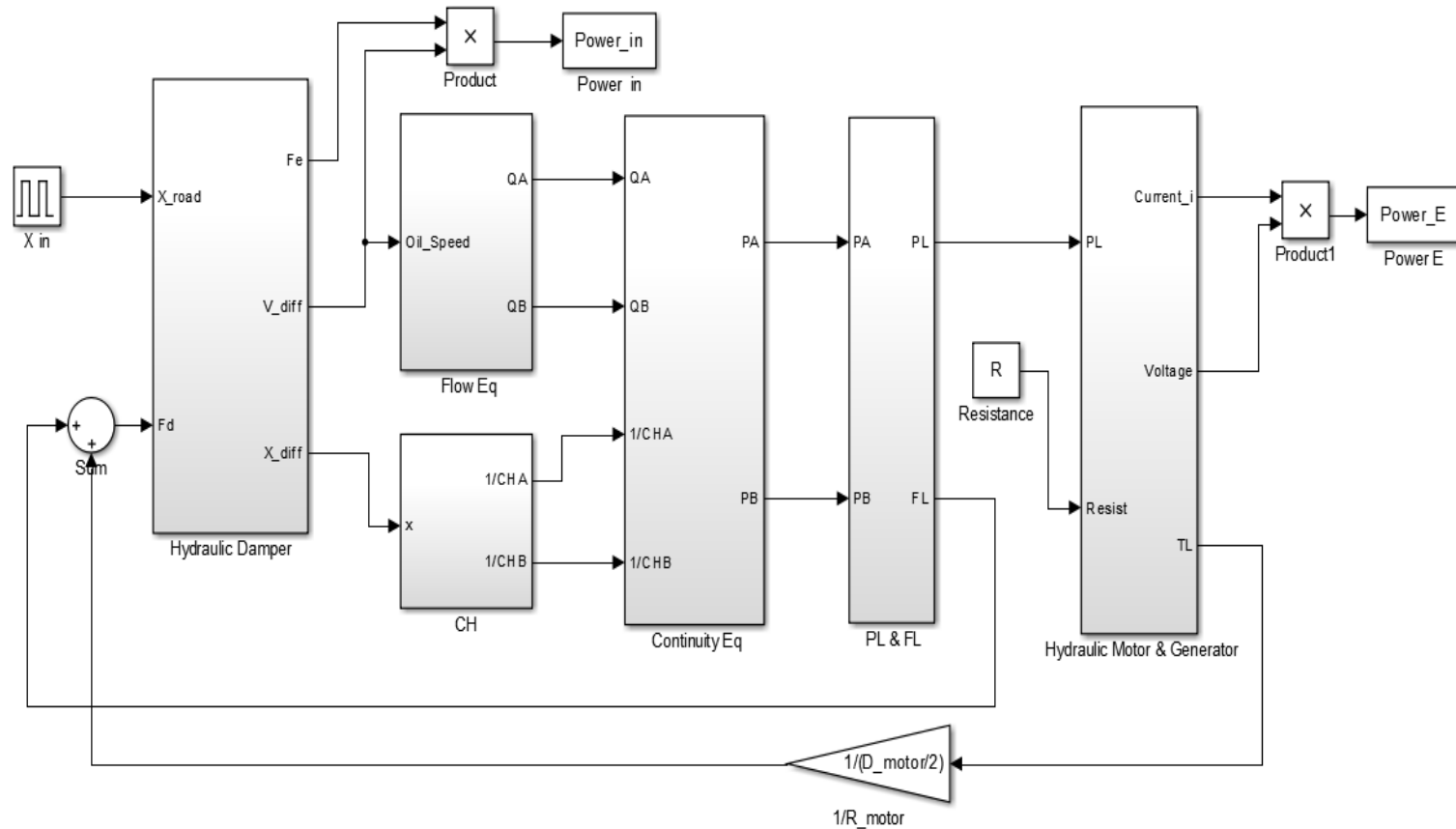


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Fig. 13



System Modelling and Simulation

■ Quarter-car system(3/3)



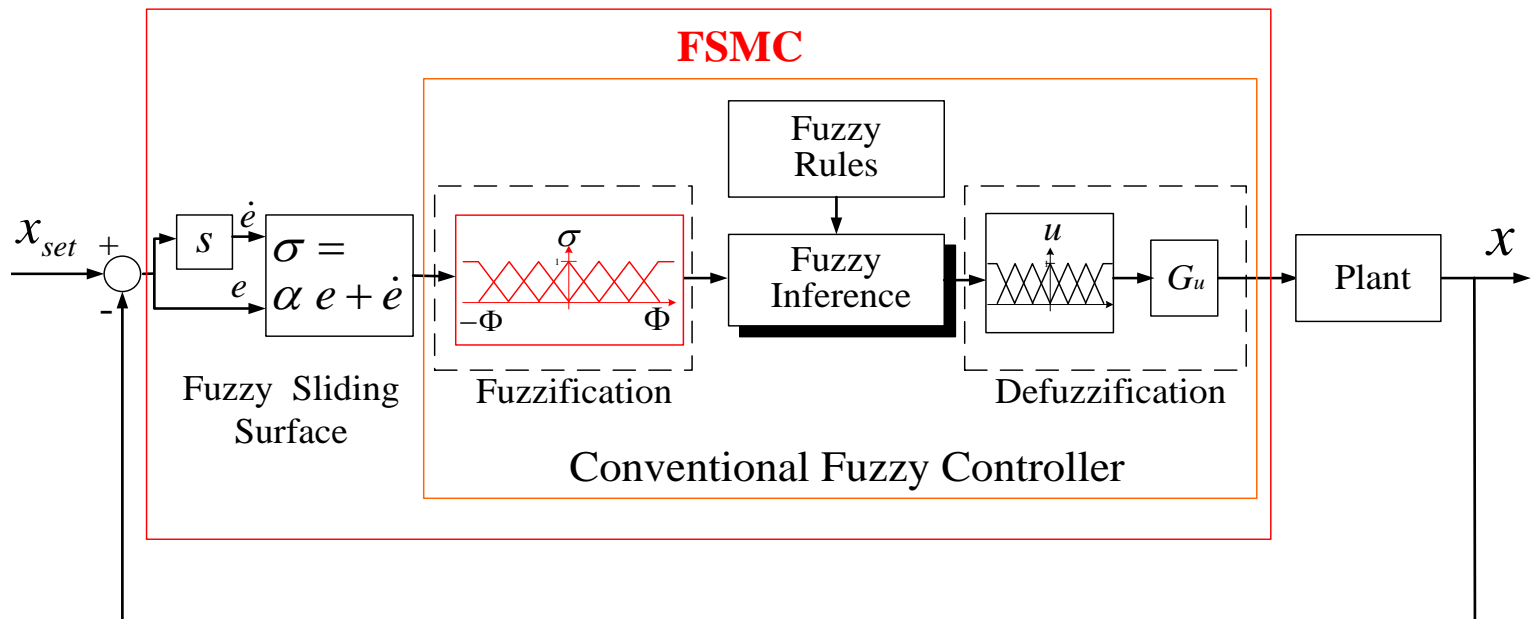
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Fuzzy Sliding Mode Control

■ Fuzzy sliding mode control



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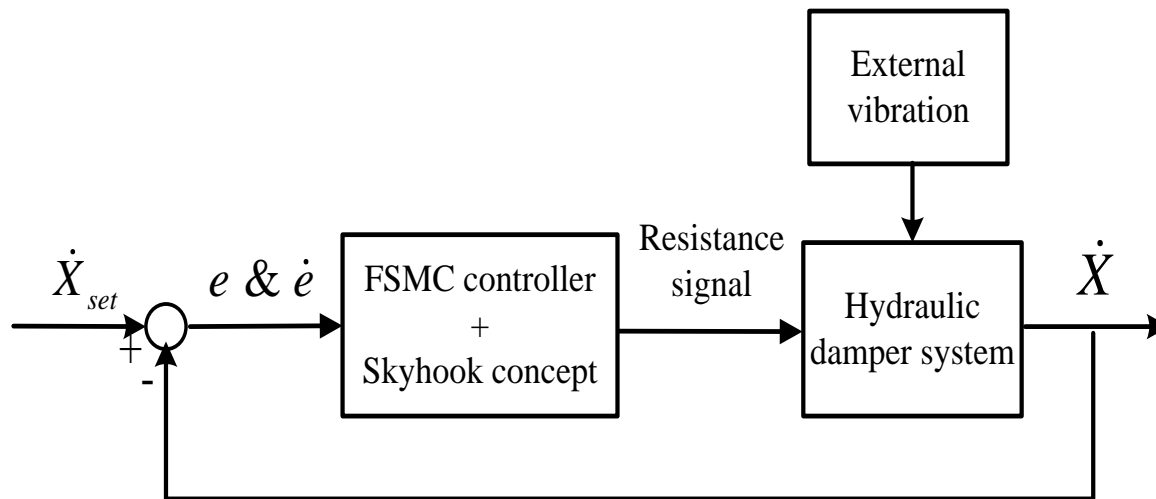
List of Experiment

5-1	Results of skyhook damper system in throttle control		
	Open-loop simulation	Open-loop experiment	
5-2	Results of skyhook damper system in resistance control		
	Open-loop simulation	Open-loop experiment	Closed-loop experiment
5-3	Results of quarter-car system in resistance control		
	Simulation	Experiment	

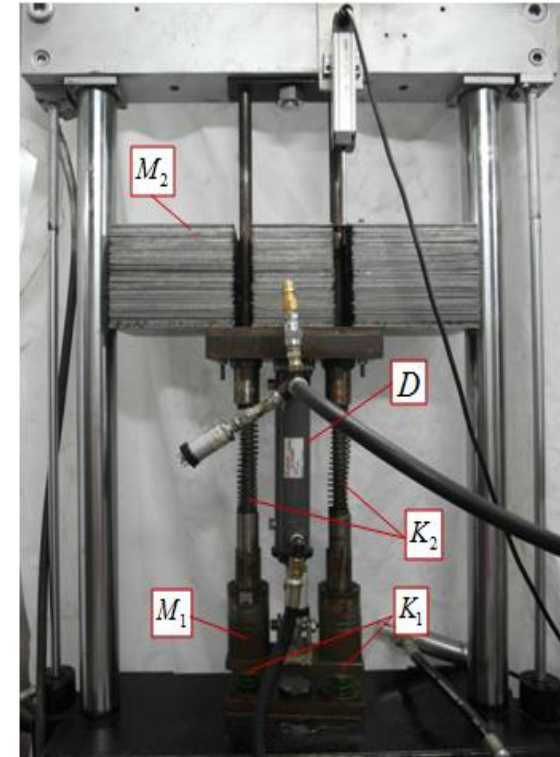


Quarter-car System in Resistance Control

■ Experiment(1/5)



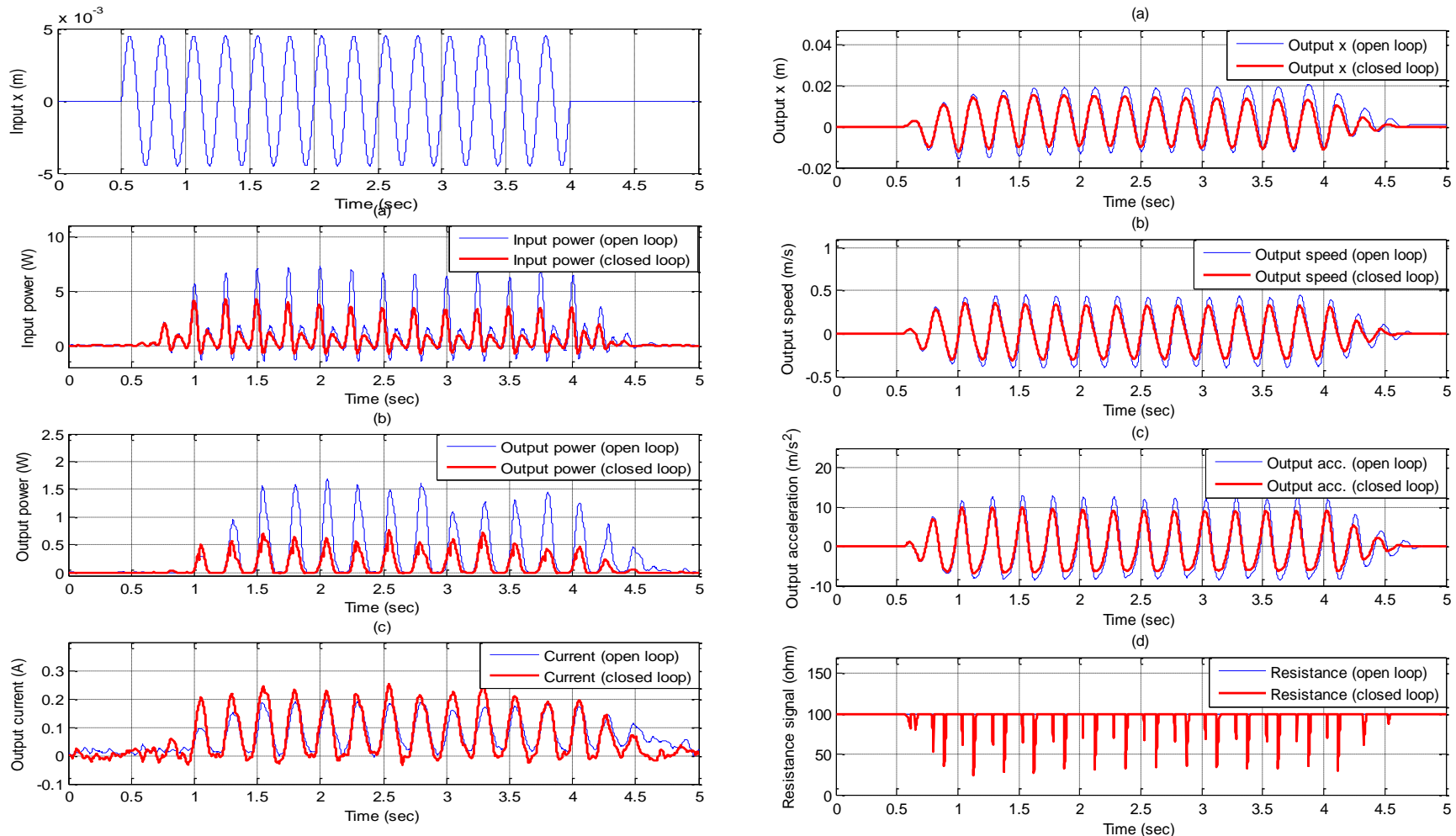
Target : Speed of $M_2 = 0$



Quarter-car System in Resistance Control

■ Experiment(2/5)

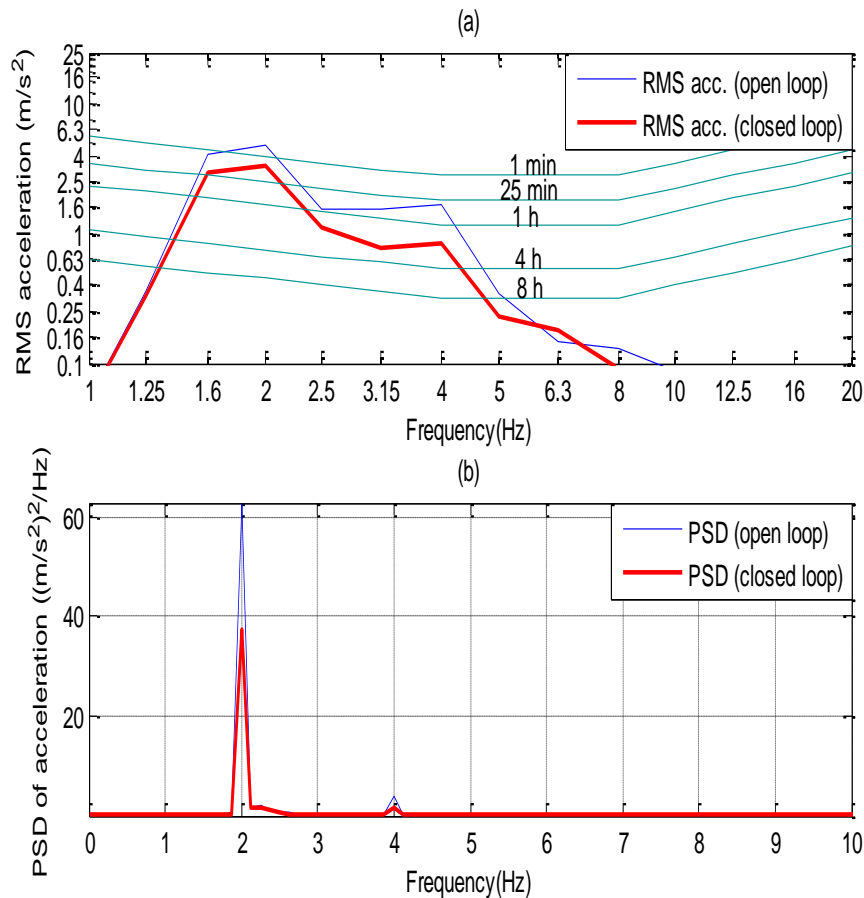
$$X(t) = 0.0045(8\pi t)m$$



Quarter-car System in Resistance Control

■ Experiment(3/5)

$$X(t) = 0.0045(8\pi t)m$$

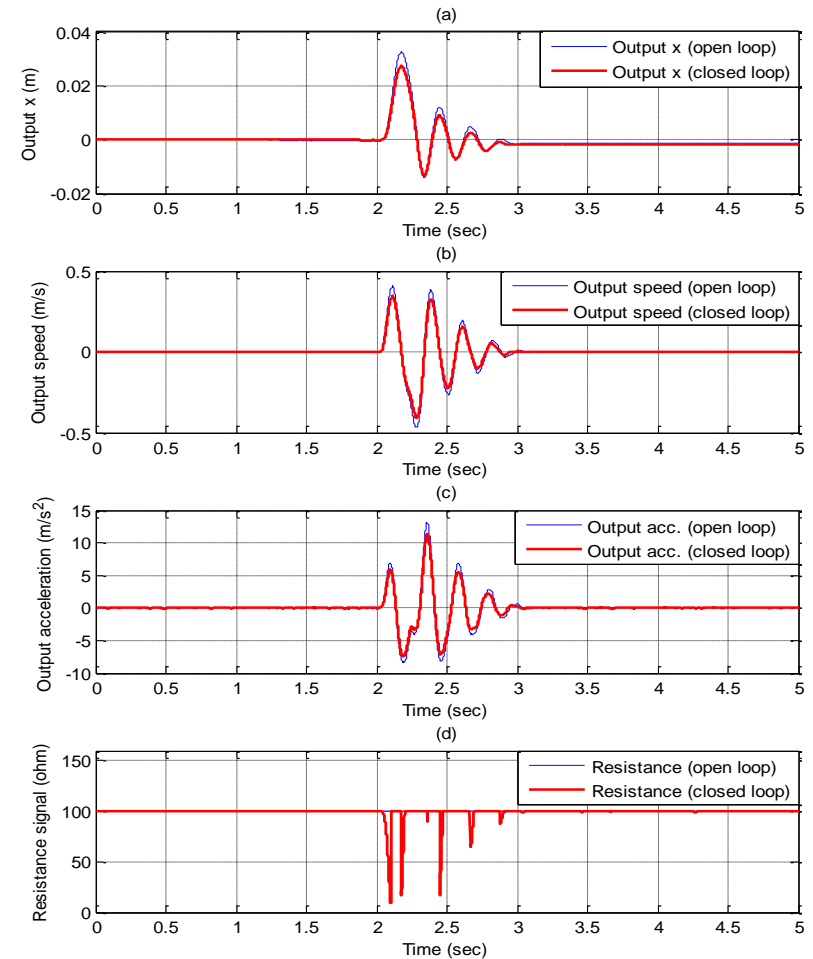
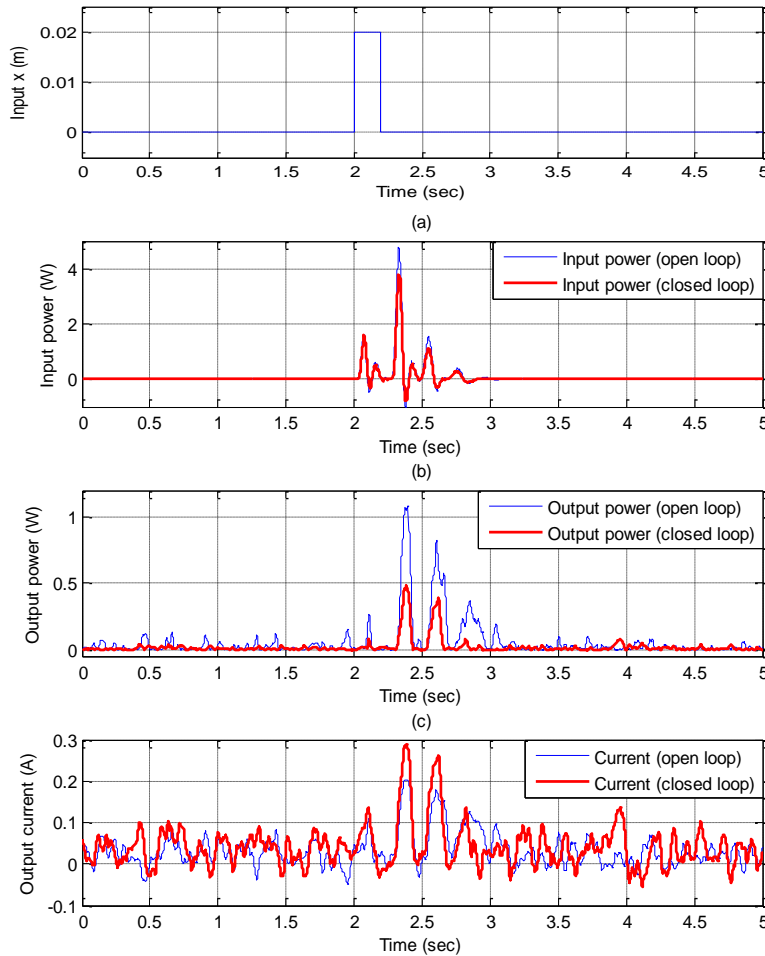


	Passive	Semi-active
Efficiency (%)	23.5%	17.3%
Car body speed (m/s)	0.42 m/s	0.37 m/s (-12%)
Acceleration (m/s ²)	12.6 m/s ²	10.1 m/s ² (-19.8%)
Current (A)	0.18 A	0.24 A
PSD _{acc,peak} ((m/s ²) ² /Hz)	63.2	37.8 (-40.2%)
Time of comfortable riding	Less than 1 min	1 min ~ 25min

Quarter-car System in Resistance Control

■ Experiment(4/5)

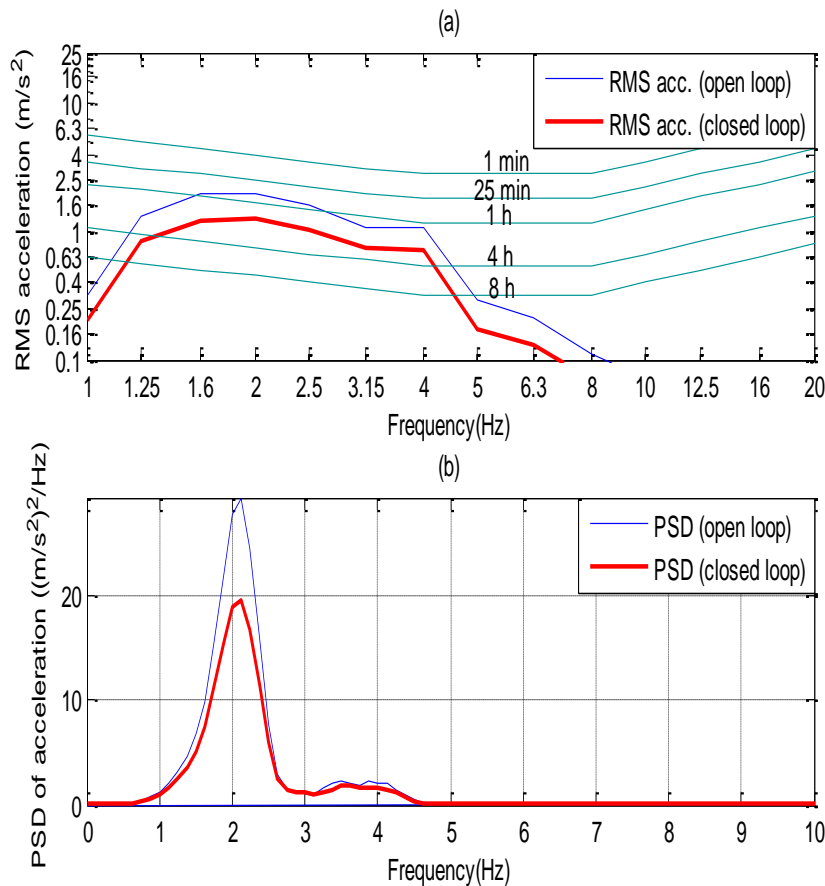
$$X(t) = 0.02m \text{ pulse}$$



Quarter-car System in Resistance Control

Experiment(5/5)

$$X(t) = 0.02m \text{ pulse}$$



	Passive	Semi-active
Efficiency (%)	24.8%	12.1%
Car body speed (m/s)	0.45 m/s	0.395 m/s (-13.3%)
Acceleration (m/s ²)	13.1 m/s ²	11.9 m/s ² (-9.2%)
Current (A)	0.2 A	0.293 A
PSD _{acc,peak} ((m/s ²) ² /Hz)	29.1	19.3 (-33.7%)
Time of comfortable riding	25min ~ 1h	1h ~ 4h

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Conclusions

■ Skyhook damper system

- The throttle valve control and the resistance control are effective to **adjust damping coefficient**.
- **Integration strategies** of energy harvesting efficiency and damping effect.
- The **efficiency** of the resistance control, 40.6%, is greater than that of the throttle control, 35.7%, due to huge oil friction in the throttle.



Conclusions

■ Skyhook damper system

- In semi-active control, **fuzzy sliding mode control** is applied. The controller can perform effectively for achieving the desired speed.
- The **ability of energy harvesting** is examined. However, when controlling the vibrational speed, the efficiency of energy harvesting will be sacrificed.



Conclusions

■ Quarter-car system

- The results have demonstrated that the speed and the PSD of acceleration are significantly **reduced**.
- The time of **comfortable riding** has been extended in Meister chart.
- The harvesting energy is little due to **light weight of the load** and limit of the test rig.



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