

# 氣壓伺服控制於高科技產業製程之應用

以 Air Bearing Conveyor Systems 應用於  
TFT-LCD製程為例

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# Outline

- Introduction
- Test Rig Layout
- Modeling of Air Bearing Conveyor System
- Controller Design
- Simulations and Experiments
- Conclusions



# Introduction

- **Flat Panel displays (FPD)** have become one of the most significant industries in the world. The manufacturers of FPD concentrate especially on the countries in **east-Asia**, including **Taiwan, Japan and Korea**.
- In order to satisfy the fast development of the request of the **market** that trends toward larger and larger sizes for **TV and monitor**, **the new generations of the glass substrates** have been developed almost **every 3 years**.
- However, **the weights and the dimensions of the glass substrates** also increase such that the manufacturing equipments have to face new process challenge.

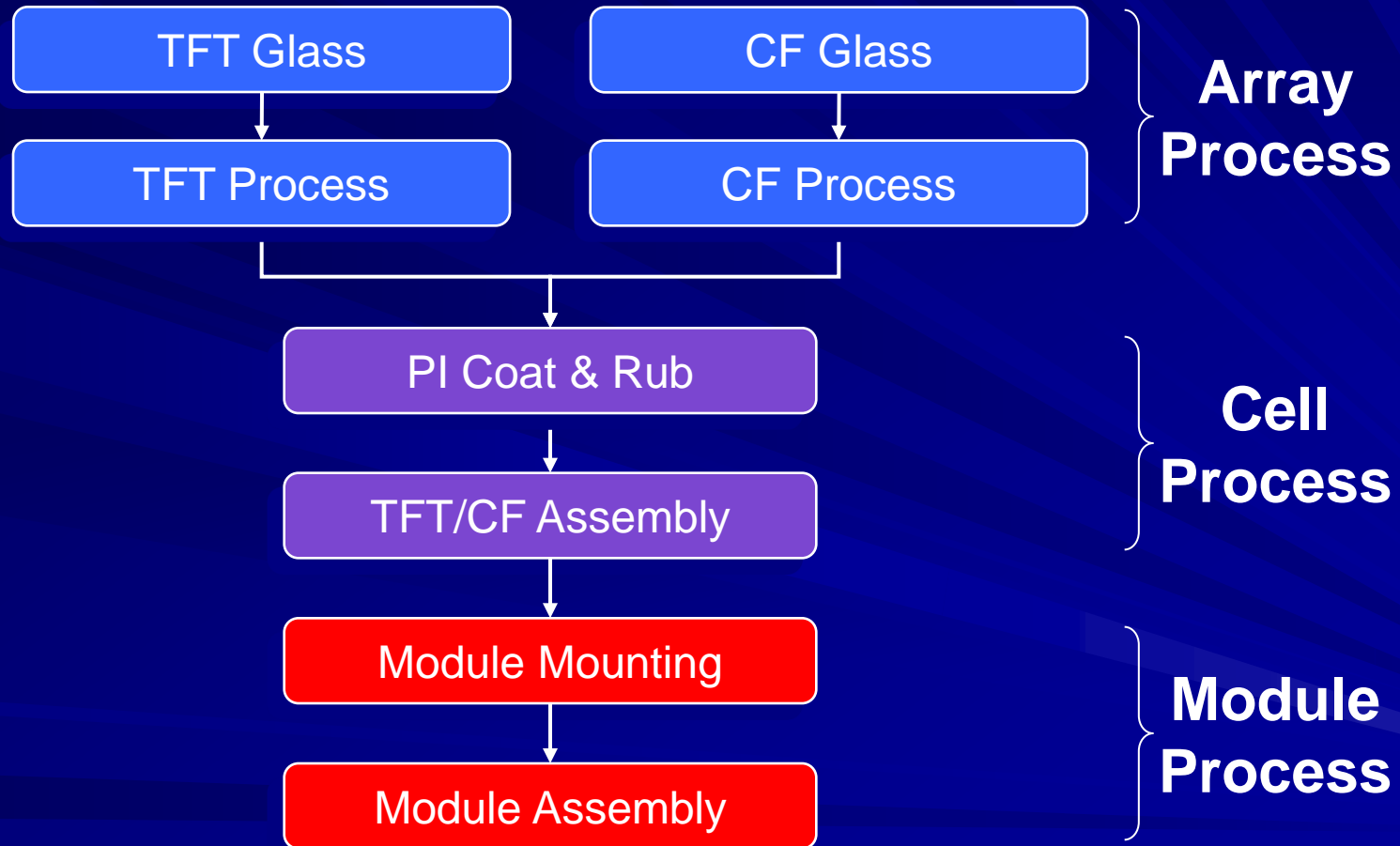


# Generation of Glass Substrates

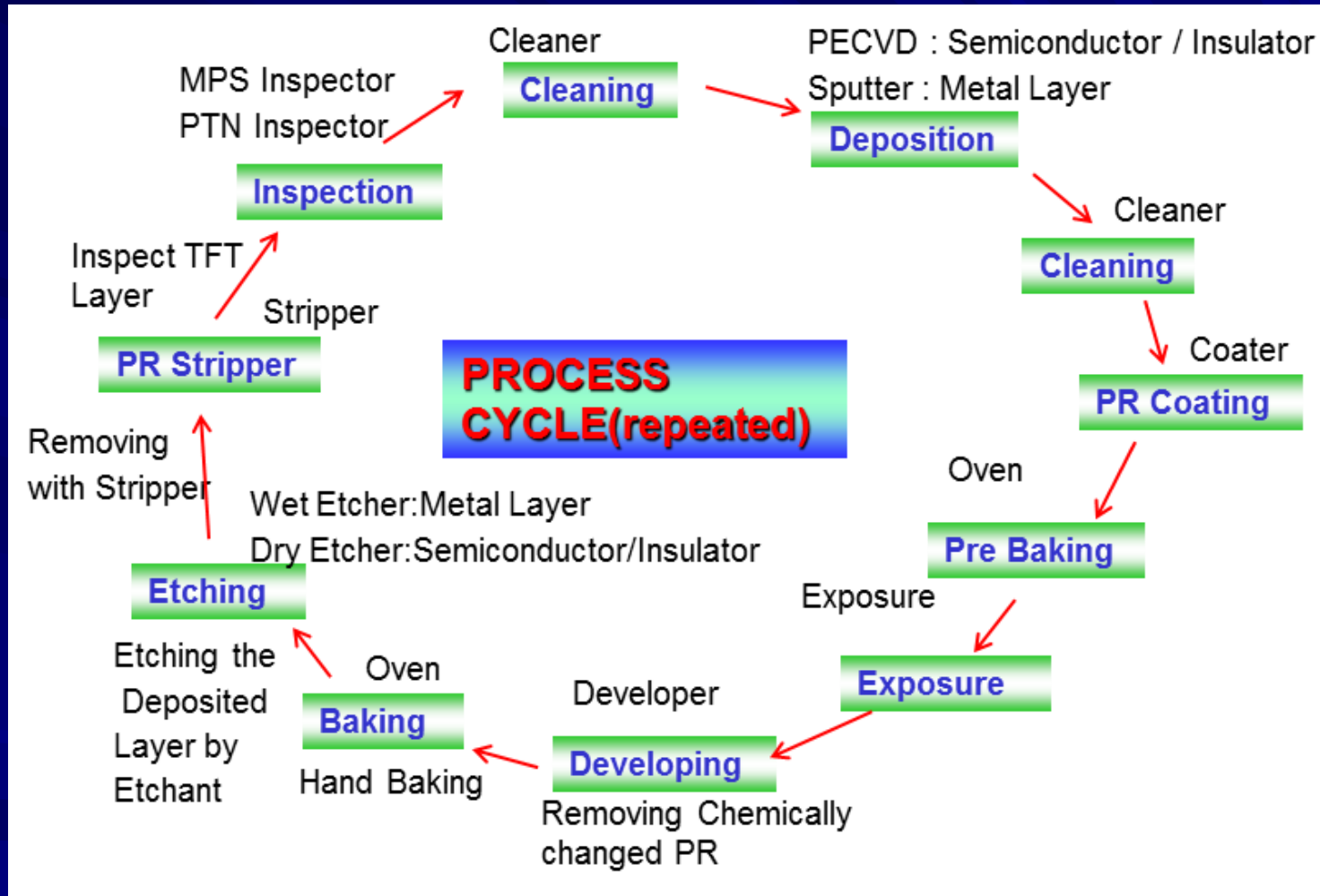
Generation	Size (mm <sup>2</sup> )	Weight (kgf)
2nd Gen	400x500	0.4
3rd Gen	550x670	0.7
3.5th Gen	600x720	0.8
4th Gen	680x880	1.1
5th Gen	1100x1250	2.5
5.5th Gen	1300x1500	3.5
6th Gen	1500x1850	5
7th Gen	1800x2000	6.5
7.5th Gen	1950x2250	7.5
8th Gen	2160x2400	8.5



# Process of TFT-LCD



# Array Process of TFT-LCD





# Introduction

- The **conveyor systems** of the glass substrate become more and more complex due to the increase of the **dimension and weight of the glass substrate**.
- The conveyor systems of the glass substrate contain **horizontal linear conveyor, horizontal rotational conveyor and vertical lifting system**.
- The horizontal linear conveyors are driven by **rollers in the conventional conveyor system** by means of the friction force between the **glass substrates and the driving rollers**.
- However, it will result in **abrasion and streaks on the contact surfaces of the glass substrates** owing to the increasing of the weight.



# Conventional Roller Conveyor Systems for TFT LCD



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# Introduction

- **Air-bearing system** can be used to form an **air cushion** between the **glass substrates** and the **air-bearing plates** such that the glass substrates can be supported by **pressured air without contact**.
- Although the **air-bearing** can tackle the **friction force**, the **unstable leveling position** of the glass substrates occur due to **unstable air pressure** between the air bearing plates and the glass substrates.
- However, some applications in the process of TFT-LCD, such as **inspection**, request the glass substrates for **keeping stable in the vertical direction**.



# Air-Bearing Conveyor Systems for TFT-LCD

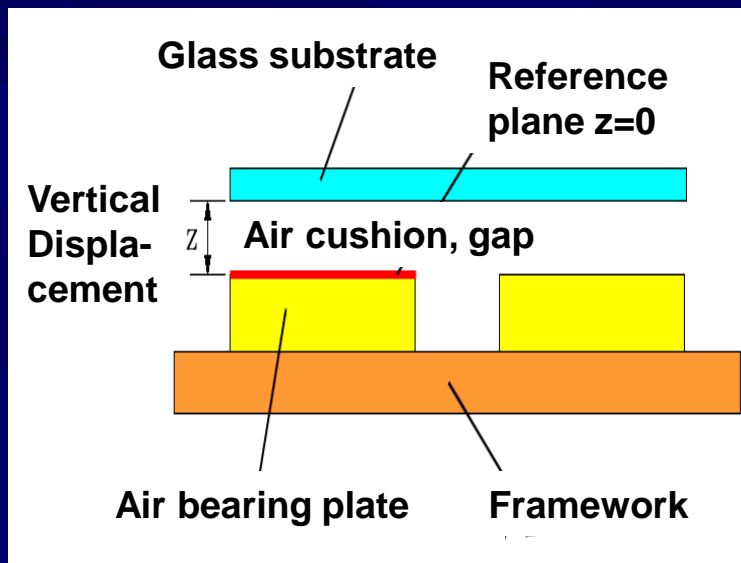
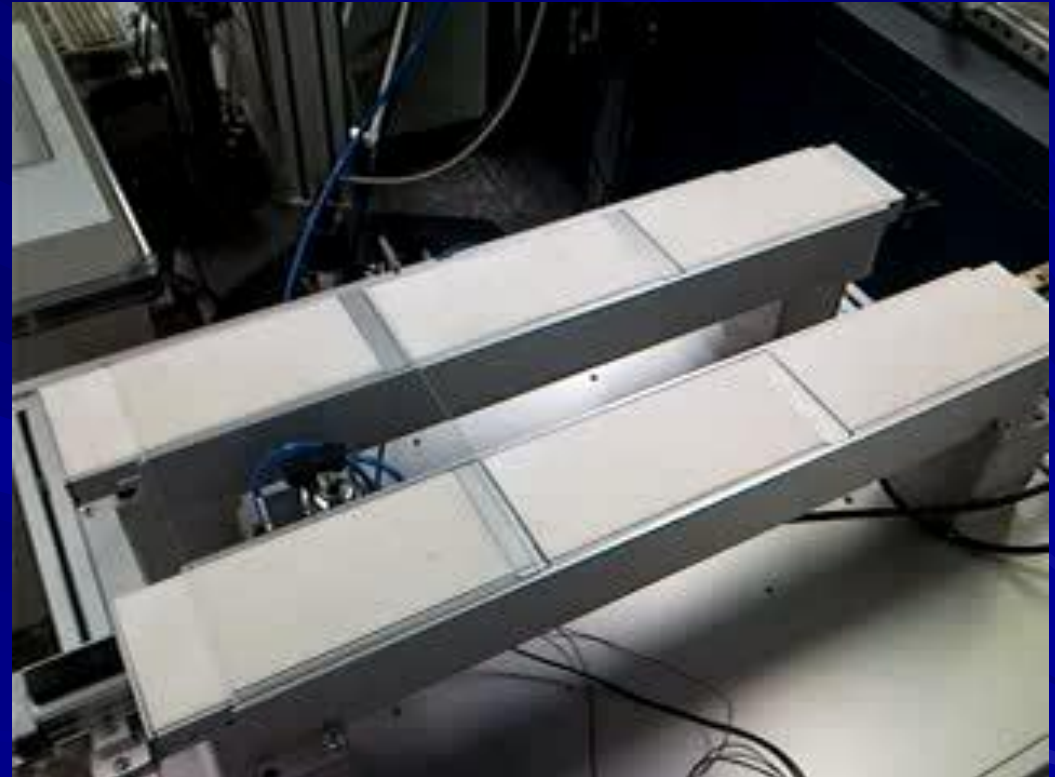


Illustration of structure of air-bearing plates



# Objective of the Research

- Active leveling control of an air bearing conveyor system for glass substrates of TFT-LCD or solar cell.
- Improvement of the damage due to friction or shaking on the surface of glass substrates
- Development of an active leveling control for an air bearing conveyor system
- Control of the floating altitude of the glass substrates for verifying the control performance, stability and robustness experimentally
- Setup of test rig for 3.5<sup>th</sup> gen. of glass substrates

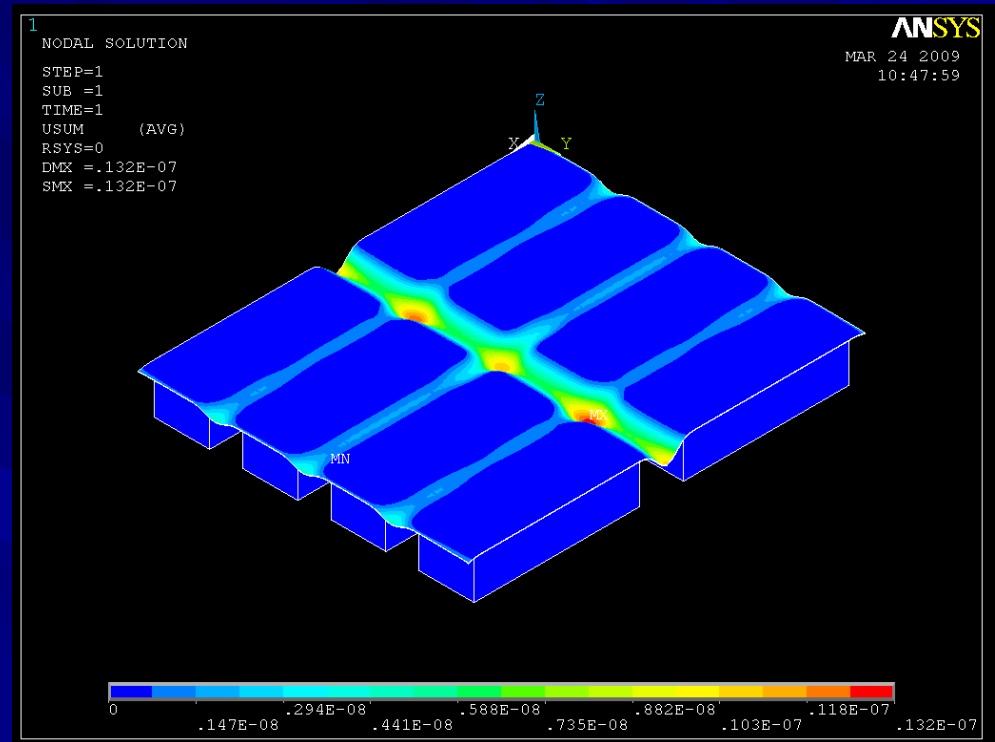
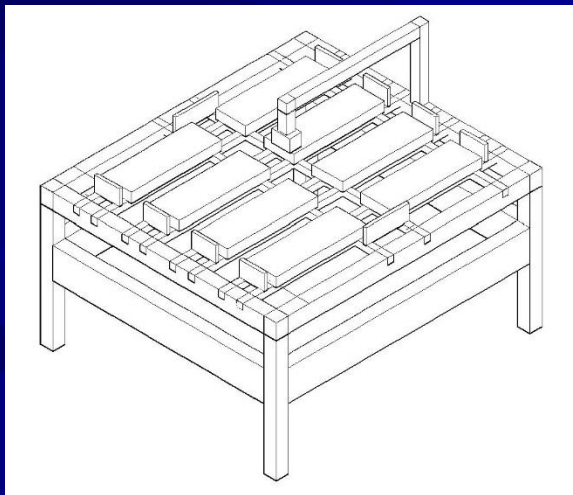
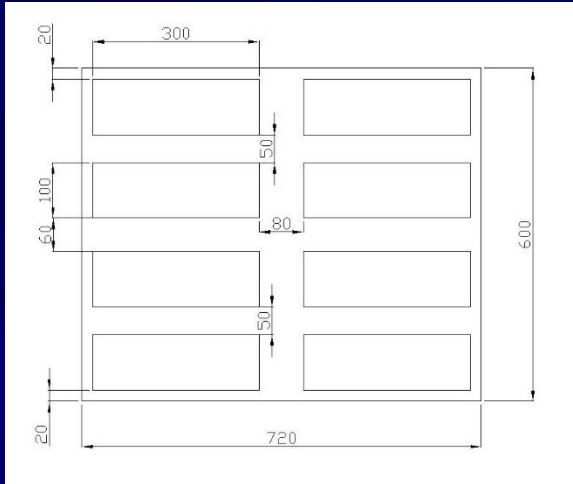


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# Deformation Analysis of Glass Substrate by ANSYS



Max. deformation  $1.32 \times 10^{-8} m$



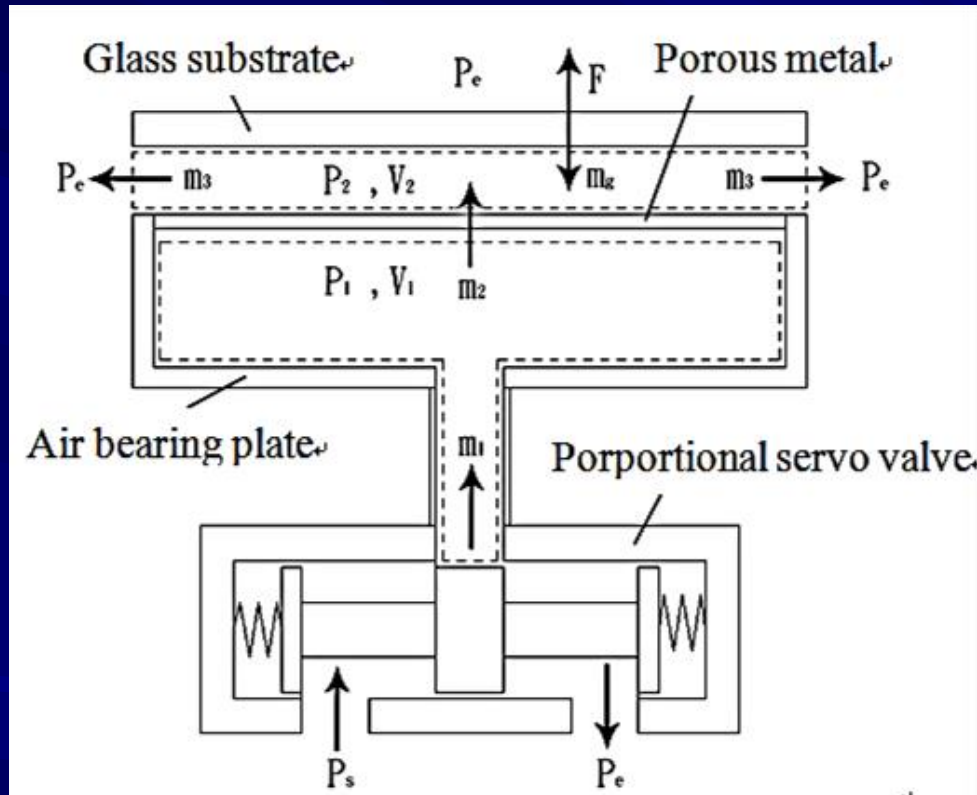


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# Illustration of Air bearing Conveyor System



$P_e$  : atmosphere pressure (1 bar)

$P_s$  : supplied pressure (5 bar)

$P_1$  : pressure in the air bearing plate

$P_2$  : pressure in the gap

$m_1$  : air mass flow (valve  $\rightarrow$  air bearing plate)

$m_2$  : air mass flow (air bearing  $\rightarrow$  gap)

$m_3$  : air mass flow (gap  $\rightarrow$  atmosphere)

$M_g$  : weight of glass substrate (N)

$V_1$  : internal volume of the air bearing plate

$V_2$  : volume of the gap

# Dynamic Model of the System

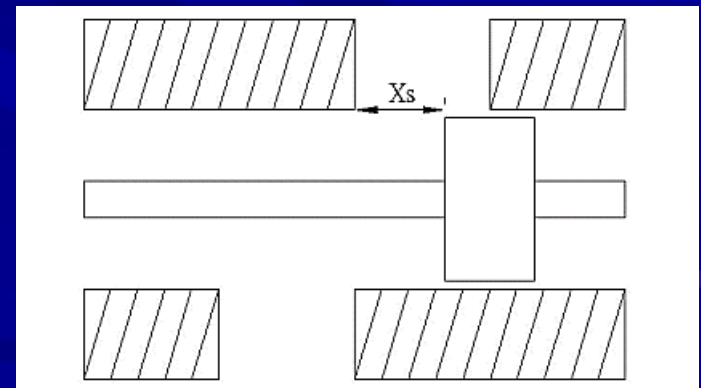
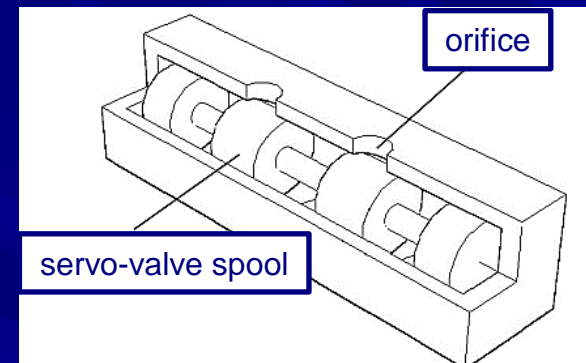
## ■ 1. Proportional Servo Valve:

$$x_s = \frac{K_s}{T_s s + 1} \cdot u_s$$

- $X_s$  : spool displacement  
 $K_s$  : constant gain of input to output  
 $T_s$  : time constant of servo valve  
 $u_s$  : voltage input

$$A_s = \begin{cases} L_s \cdot X_s & 0 \leq X_s \leq X_{\max} \\ 0 & X_{\max} \leq X_s \leq 0 \end{cases}$$

- $A_s$  : area of orifice  
 $L_s$  : outside dimension of spool  
 $X_s$  : displacement of spool  
 $X_{\max}$  : maximum displacement of spool



# Dynamic Model of the System

## ■ 2. Mass Flow Rate of Orifice:

Equilibrium equation by Mc Cloy

$$\dot{m} = C_d C_m \frac{P_{up}}{\sqrt{T_u}} A_h$$

$\dot{m}$  : mass flow rate of orifice

$C_d$  : output/displacement flow rate

$C_m$  : parameter of mass flow rate

$P_{up}$  : Stagnation pressure of upstream

$T_{up}$  : Stagnation temperature of upstream



# Dynamic Model of the System

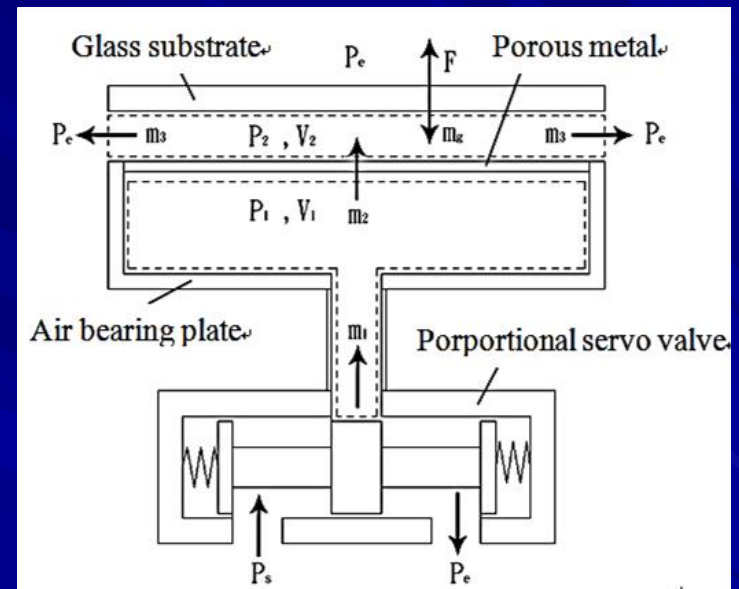
## ■ 3. Continuous equation

pressure rate in the air bearing plate

$$\dot{P}_1 = \frac{\gamma}{V_1} \cdot (RT_1(\dot{m}_1 - \dot{m}_2))$$

pressure rate in the gap

$$\dot{P}_2 = \frac{\gamma}{V_{2D} + A_a \cdot Y} \cdot (RT_2(\dot{m}_2 - \dot{m}_3) + A_a P_2 \dot{Y})$$

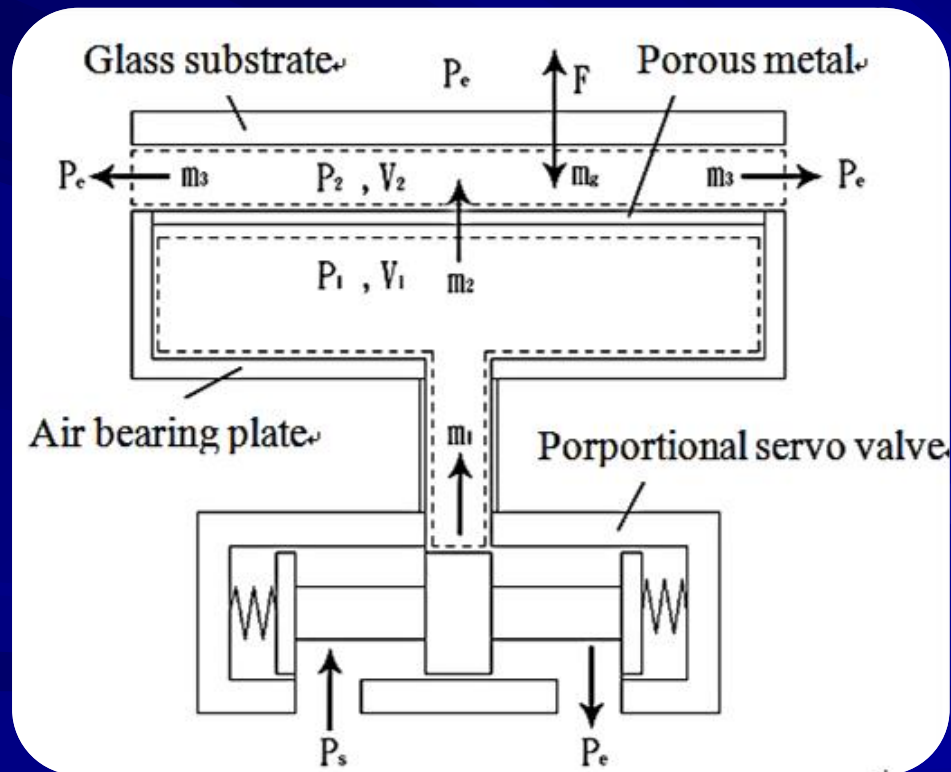




# Dynamic Model of the System

## ■ 4. Motion equation

$$M \cdot \ddot{Y} = (P_2 - P_e) \cdot A_a - M_g - F_r$$



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# Controller Design

- Pneumatic system is a **nonlinear time-varying system**, so It's difficult to build up an accurate mathematical model and get the accurate parameters.
- In this research, we use functional approximation with Fourier series based to build up an adaptive sliding mode control, **Fourier series-based adaptive sliding mode controller (FSB-ASMC)**.
- **FSB-ASMC with  $H_\infty$  tracking performance** is using to handle the function approximation errors, un-modeled dynamics and disturbances.



# Controller Design

Functional Approximation with Fourier Series

+

Sliding Mode Controller

+

$H^\infty$  Tracking Performance



**FSB-ASMC+ $H^\infty$  Controller**

**Model-Free, Robustness**



# Controller Design

In order to develop the controller for the pneumatic air-bearing servo system, FSB-ASMC+H $\infty$  controller is used in this study to solve the high non-linearity and time-varying problems.

**STEP1:** A general nonlinear system is shown as

$$y^{(n)}(t) = f(\mathbf{x}, t) + g(t)u(t) + d(t) = F(t) + g(t)u(t)$$

**STEP2:** Define the output error as

$$e(t) = y(t) - y_m(t)$$

**STEP3:** Define the sliding surface

$$s = a_1 e(t) + a_2 \dot{e}(t) + \dots + e^{(n-1)}(t)$$

**STEP4:** Choose the controller input

$$u(t) = \frac{-\hat{\mathbf{W}}_F^T \mathbf{q}_F(t) - a_1 \dot{e} - a_2 \ddot{e} - p_{21} e - p_{22} \dot{e} + y_m^{(3)}(t) - \frac{s}{2\rho^2}}{\hat{\mathbf{W}}_g^T \mathbf{q}_g(t)}$$



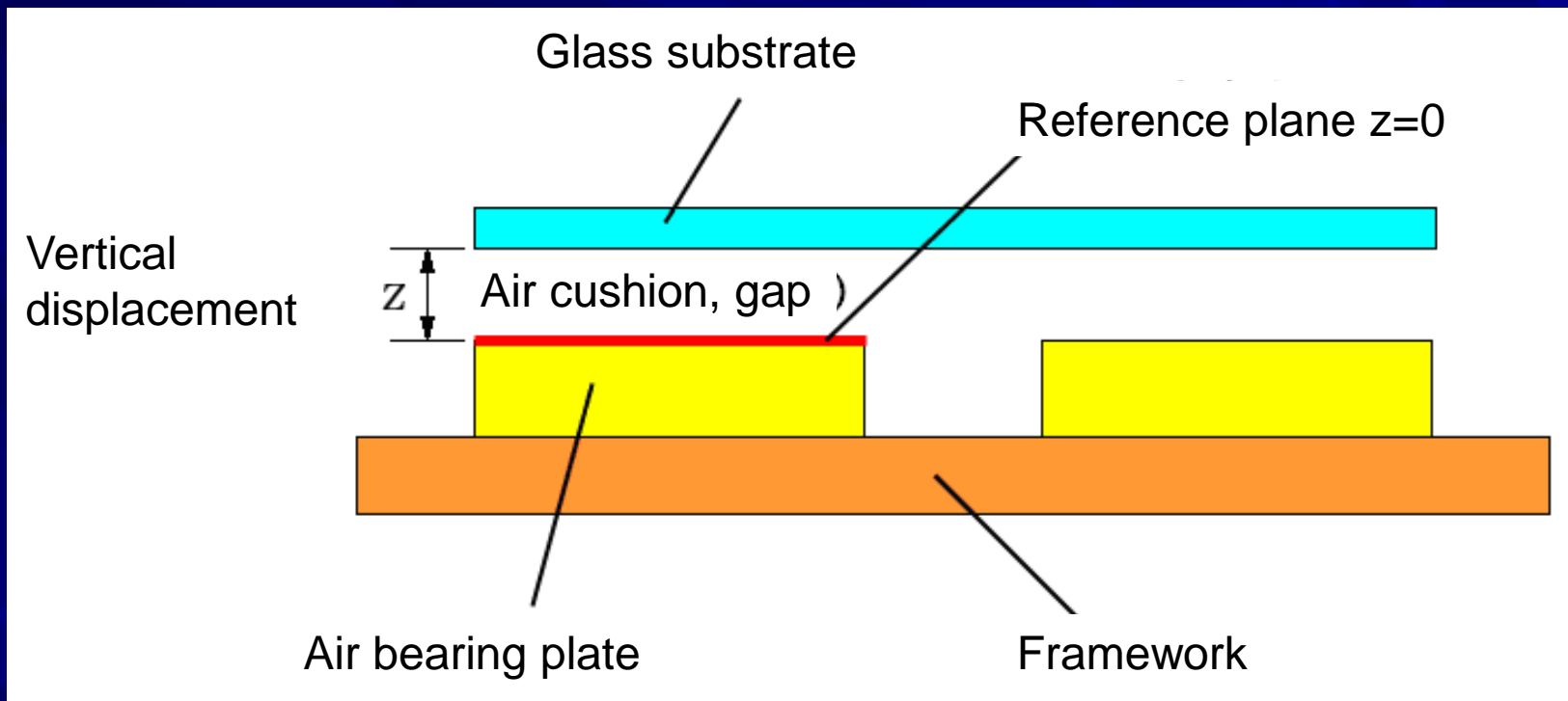


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# Simulation of Vertical Displacement of Glass Substrate



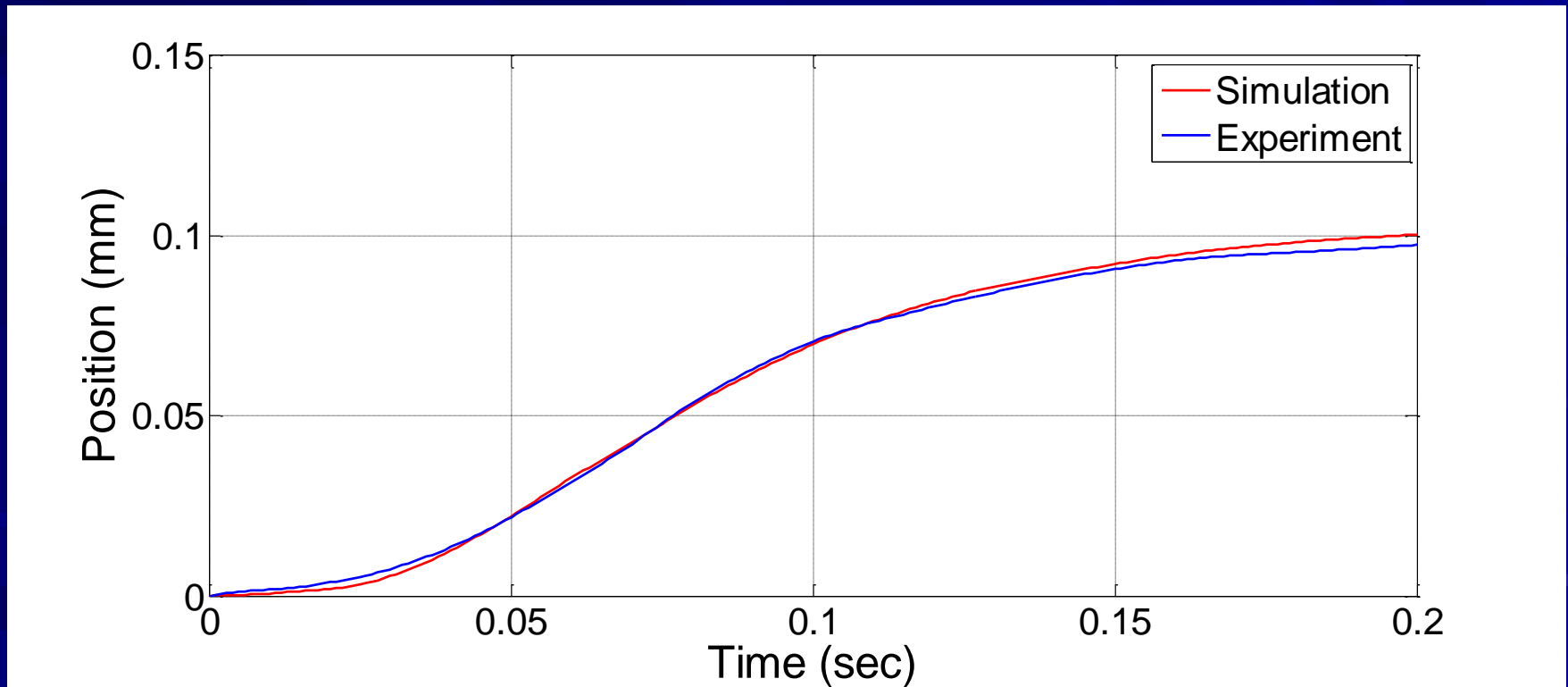
# Simulation of Displacement of Glass Substrate in Open-Loop

- Comparison between simulation and experiment in open loop
- Oscillation occurs without controller
- Step response in closed loop
- 5<sup>th</sup> order position tracking in closed loop
- 5<sup>th</sup> order tracking control + position control in closed loop
- Sinusoid position tracking in closed loop



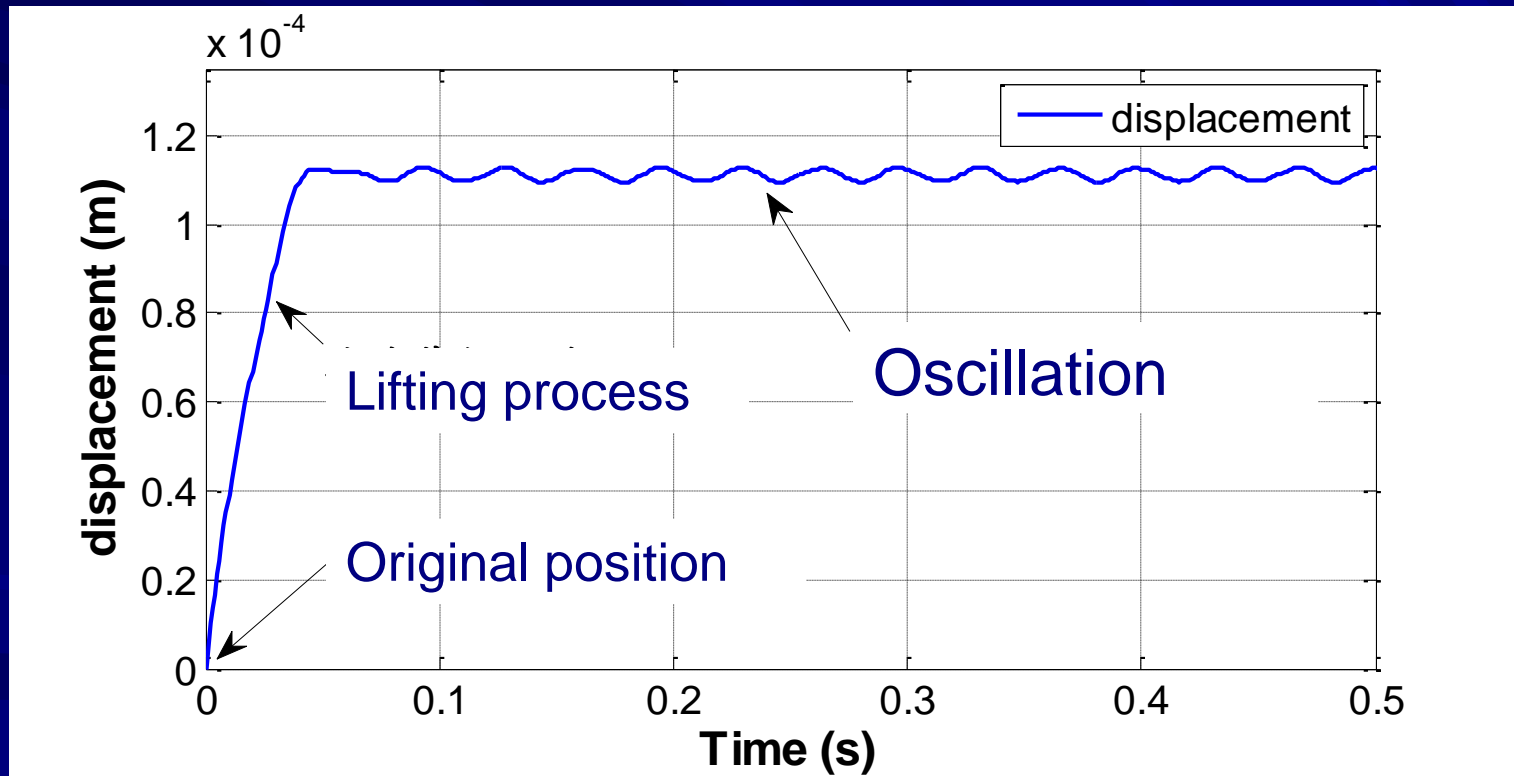
# Simulation of Displacement of Glass Substrate in Open-Loop

- Comparison between simulation and experiment in open loop system



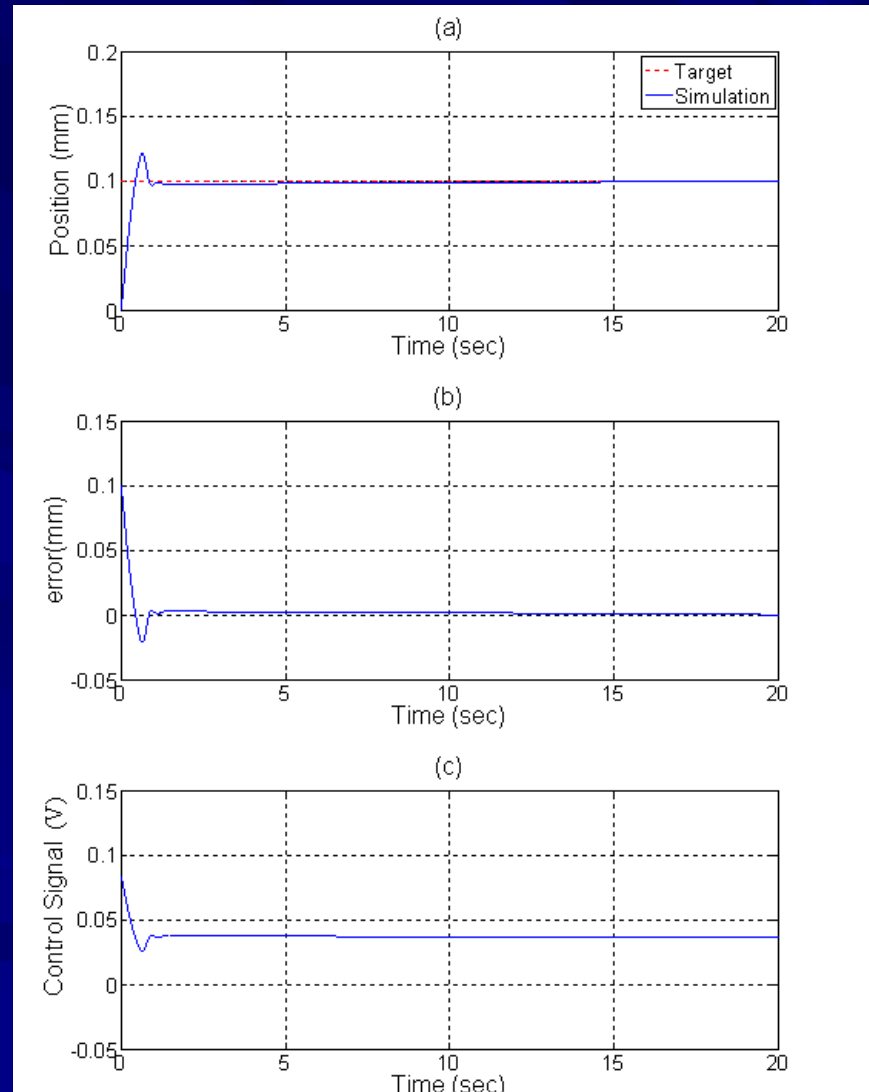
# Simulation of Displacement of Glass Substrate in Open-Loop

- Oscillation occurs when the glass substrate reaches the stable height without controller.





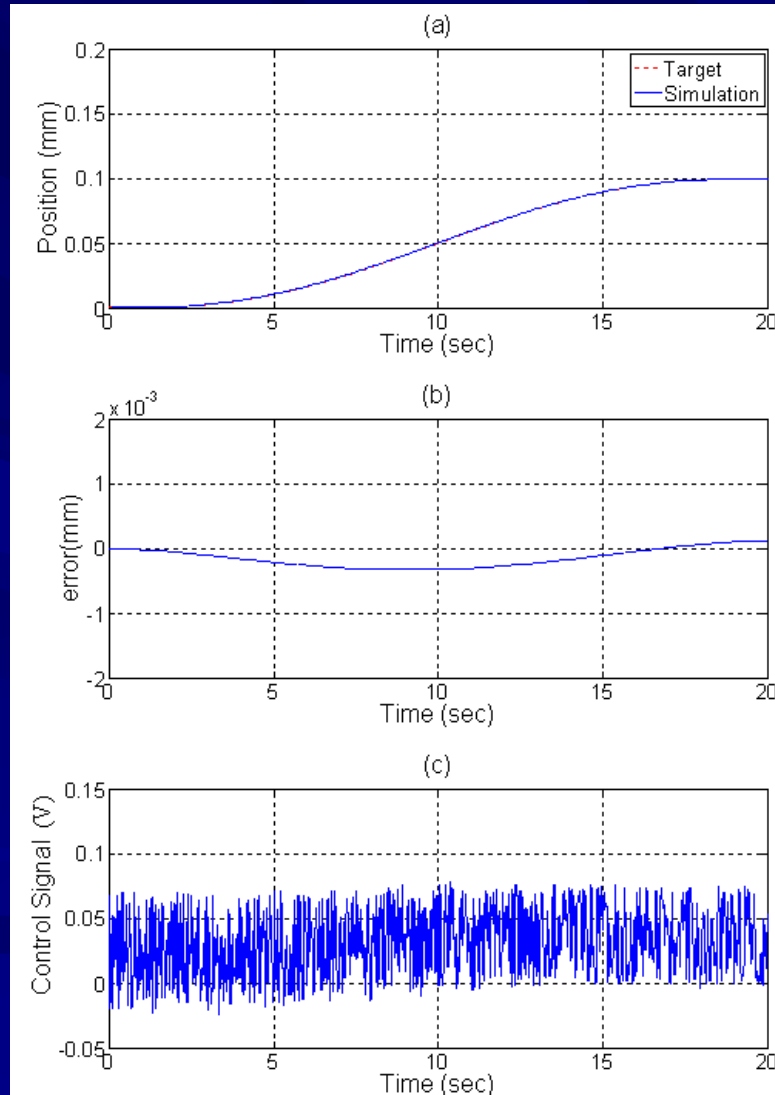
# Simulation of step response in closed loop (stroke: 0.1 mm, 20 sec)



(a) trajectory response  
(b) error  
(c) control signal



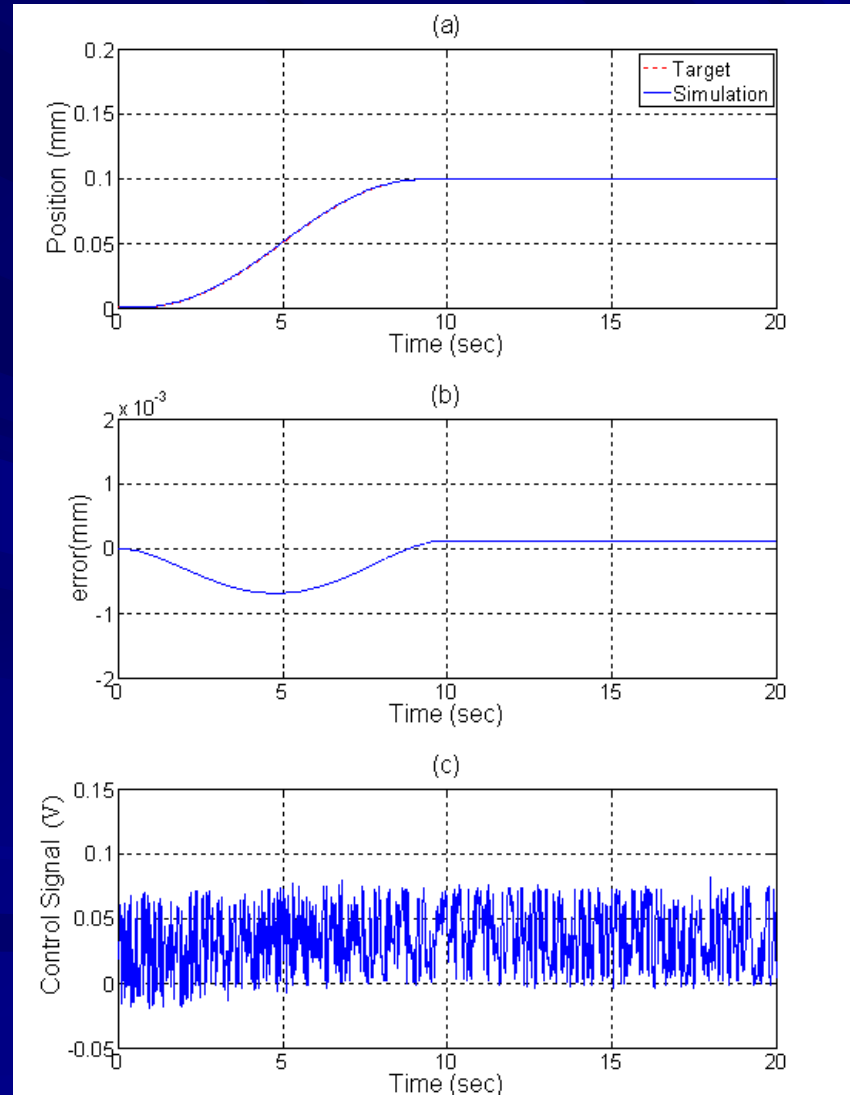
# Simulation of 5<sup>th</sup> order position tracking in closed loop (stroke: 0.1 mm, 20 sec)



- (a) trajectory response
- (b) error
- (c) control signal



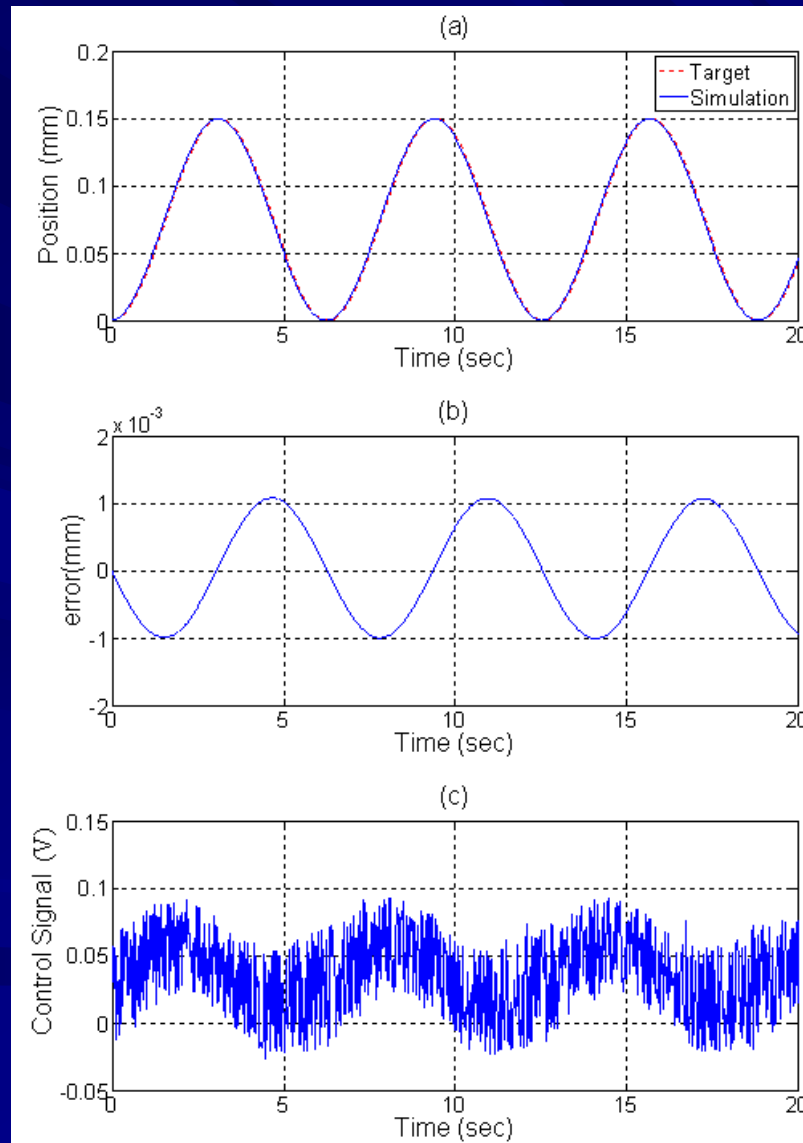
# Simulation of 5<sup>th</sup> order tracking control + position control in closed loop (stroke: 0.1 mm, 20 sec)



(a) trajectory response  
(b) error  
(c) control signal



# Simulation of sine position tracking in closed loop (stroke: 0.075 mm, $2\pi$ sec/period, 20 sec)



- (a) trajectory response
- (b) error
- (c) control signal

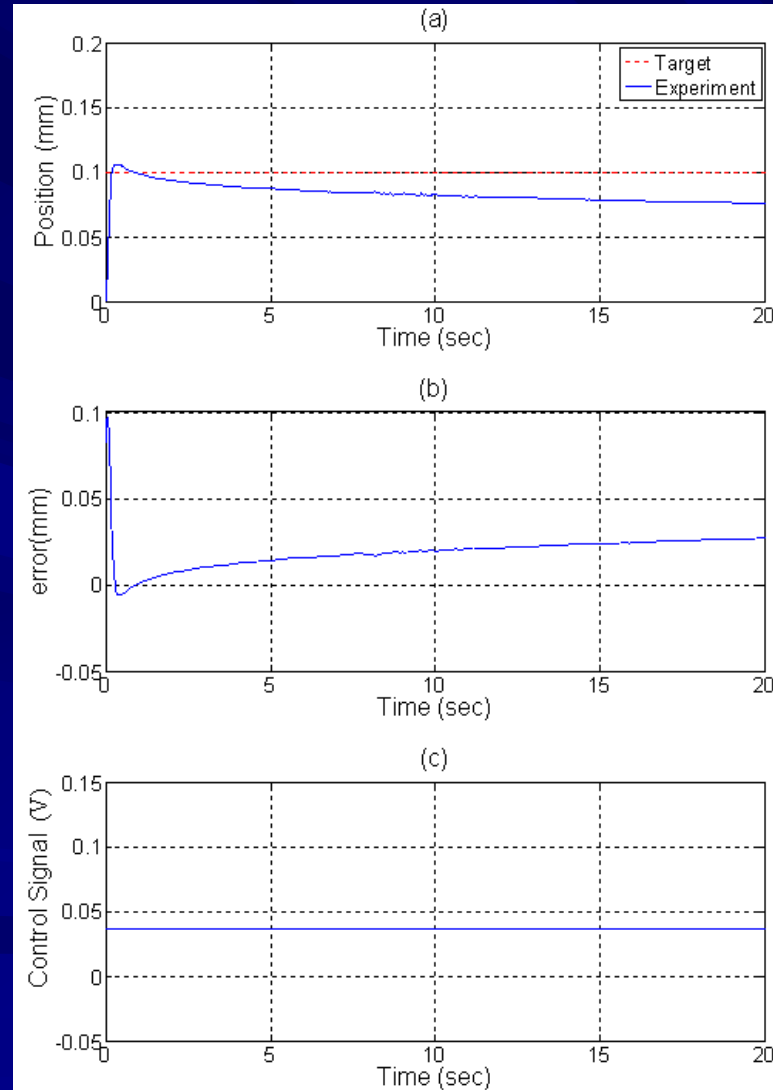


# Experiment of Displacement of Glass Substrate in Open-Loop

- Experiment in open loop
- Open loop with disturbance
- Step response in closed loop
- 5<sup>th</sup> order position tracking in closed loop
- 5<sup>th</sup> order tracking control + position control in closed loop
- Sinusoid position tracking in closed loop



# Experiment of displacement of glass substrate in open loop (stroke: 0.1 mm, 20 sec)

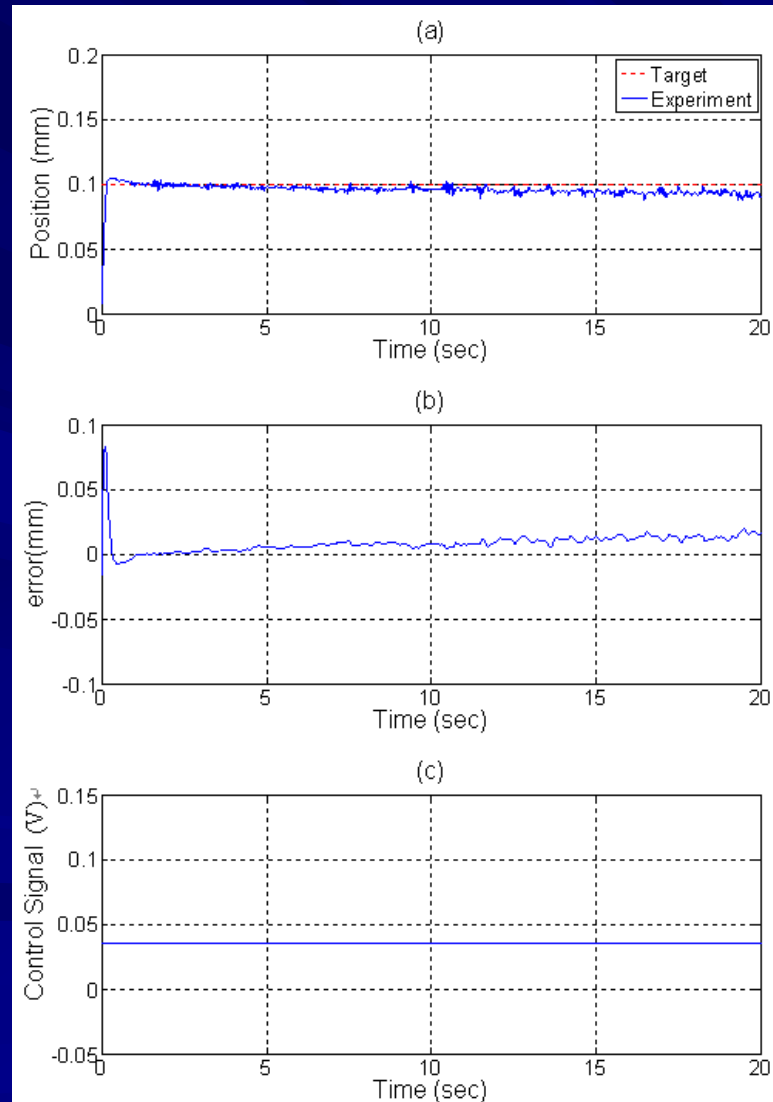


- (a) trajectory response
- (b) error
- (c) control signal





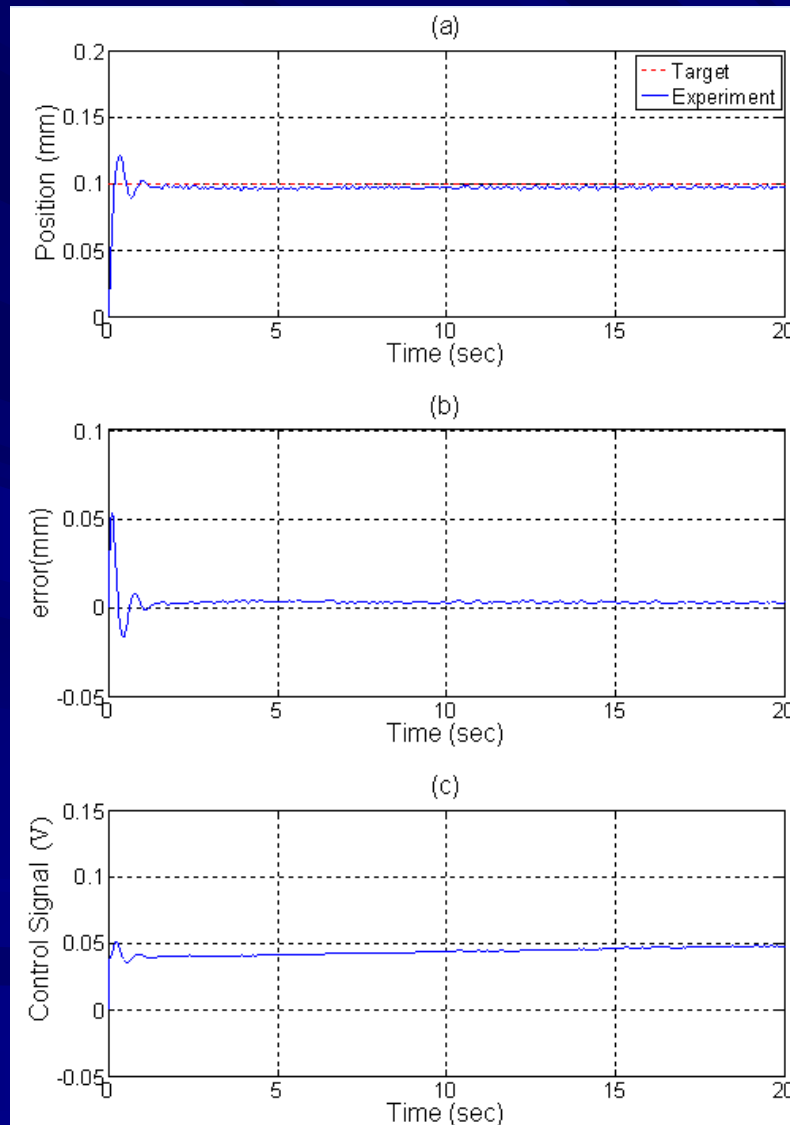
# Experiment of displacement variation of glass substrate in open loop with disturbance (stroke: 0.1 mm, 20 sec)



- (a) trajectory response
- (b) error
- (c) control signal



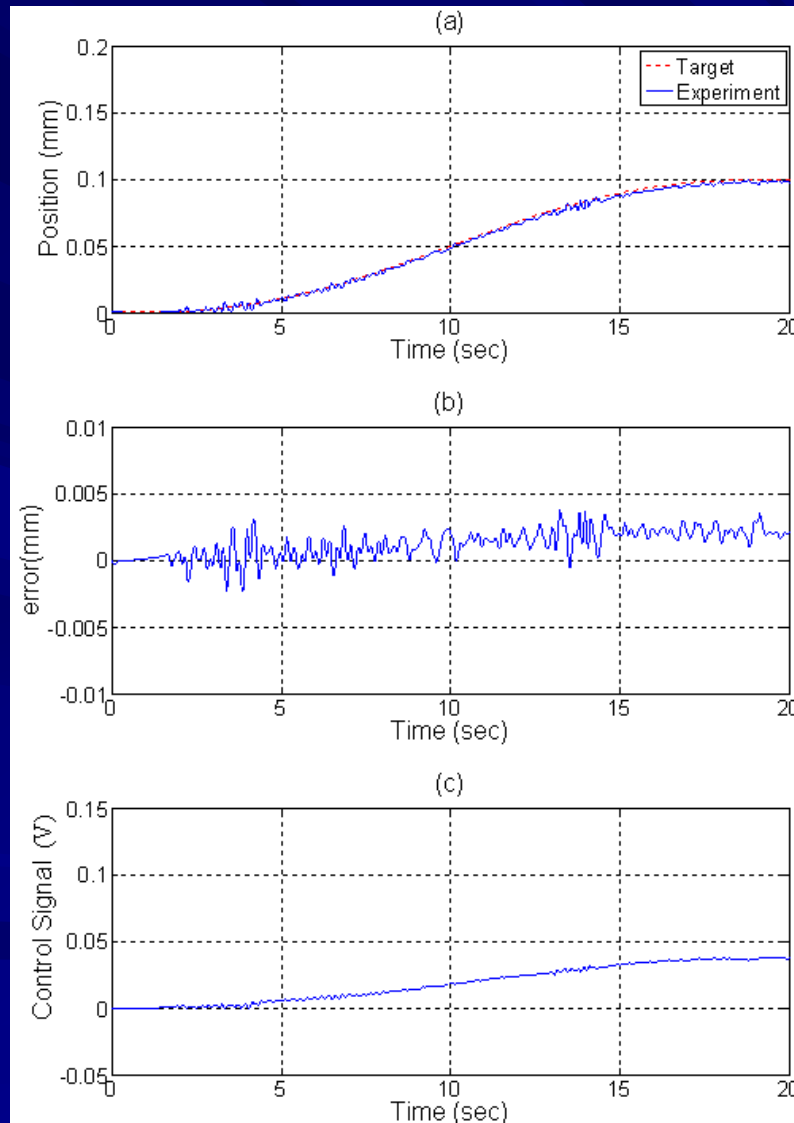
# Experiment of step response in closed loop (stroke: 0.1 mm, 20 sec)



- (a) trajectory response
- (b) error
- (c) control signal



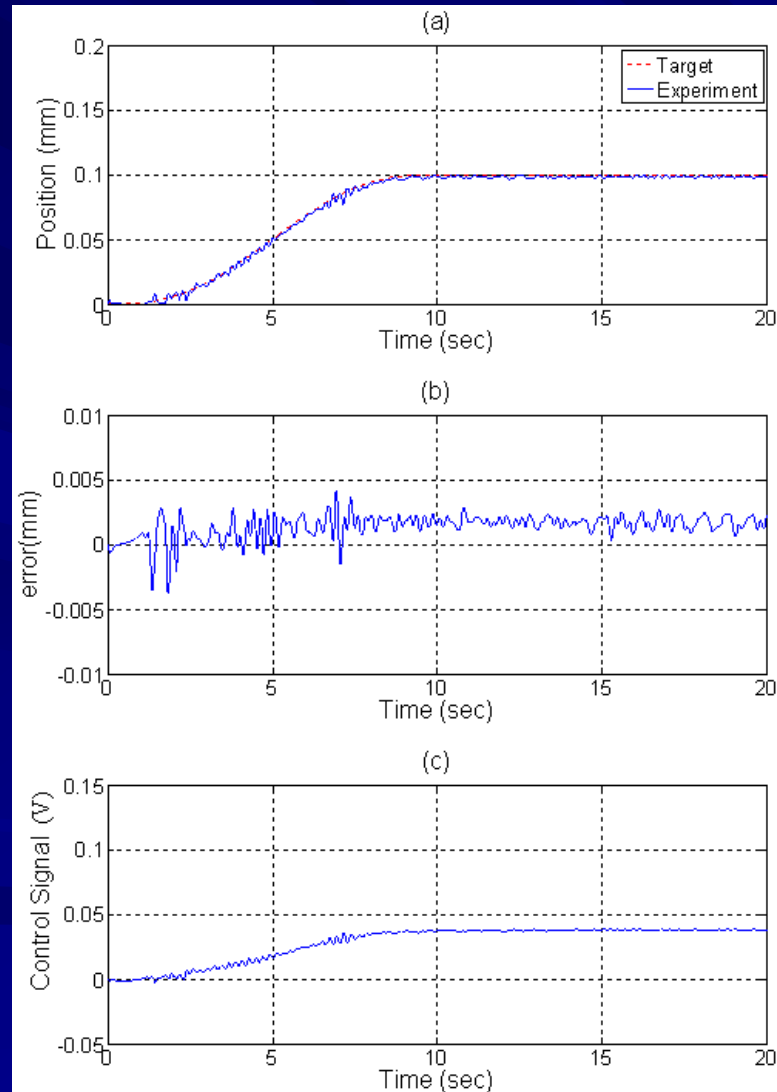
# Experiment of 5<sup>th</sup> order tracking control in closed loop (stroke: 0.1 mm, 20 sec)



(a) trajectory response  
(b) error  
(c) control signal



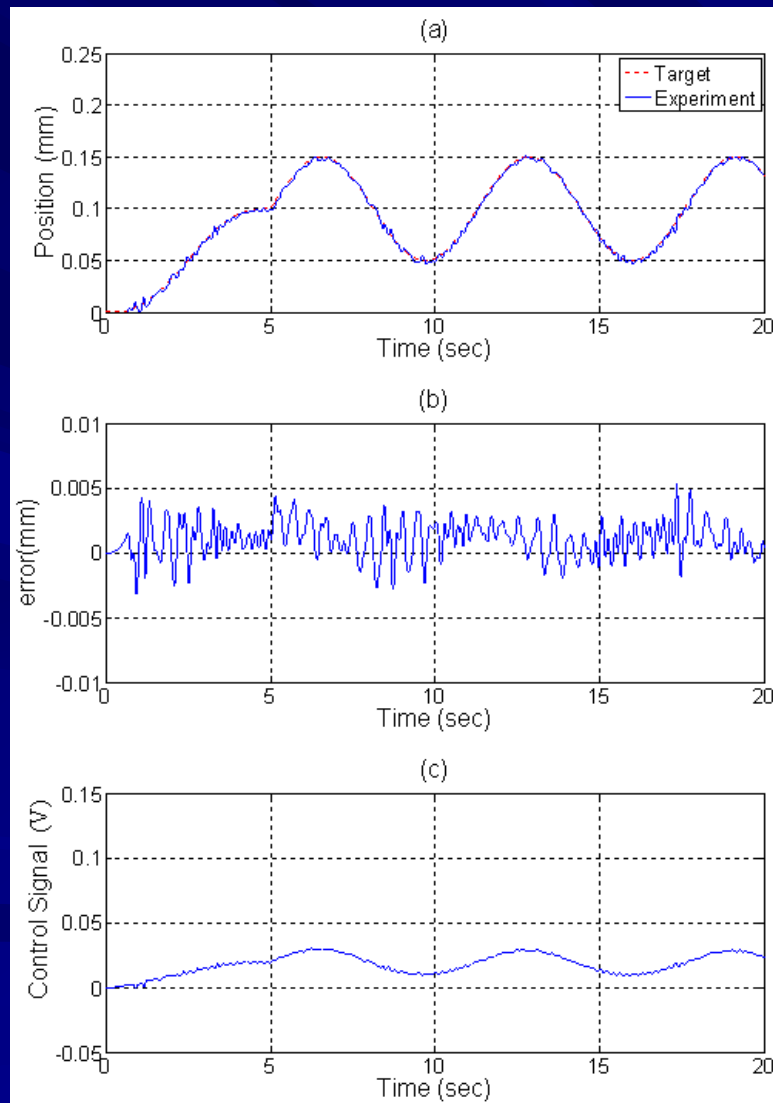
# Experiment of 5<sup>th</sup> order tracking control + position control in closed loop (stroke: 0.1 mm, 20 sec)



- (a) trajectory response
- (b) error
- (c) control signal



# Experiment of sine tracking control in closed loop (stroke: 0.05 mm, $2\pi$ sec/period, 20 sec)



(a) trajectory response  
(b) error  
(c) control signal



# Comparison and Discussion of Experiment Results

- Comparison of open-loop and closed-loop control
- Comparison of different controllers
- Comparison of different strokes
- Comparison of different loads
- Comparison of sine path tracking control
- Robust test





# Experimental comparison of open loop and closed loop

## Case (1)

stroke : 0.1 mm

time : 20 sec

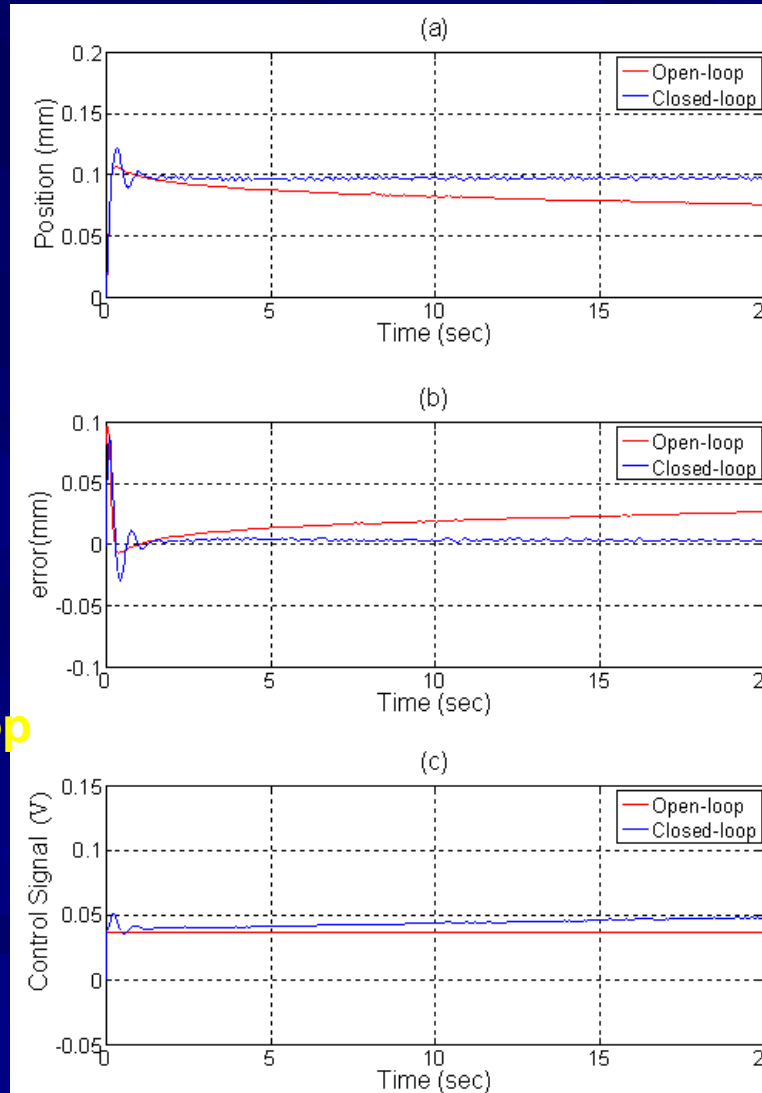
system: open loop

## Case (2)

stroke : 0.1 mm

time : 20 sec

system: closed loop



(a) trajectory response

(b) error

(c) control signal



# Experimental comparison of different controllers

## Case (1)

stroke : 0.1mm

5<sup>th</sup> order : 20sec

Time : 20sec

**System : without  $H_\infty$**

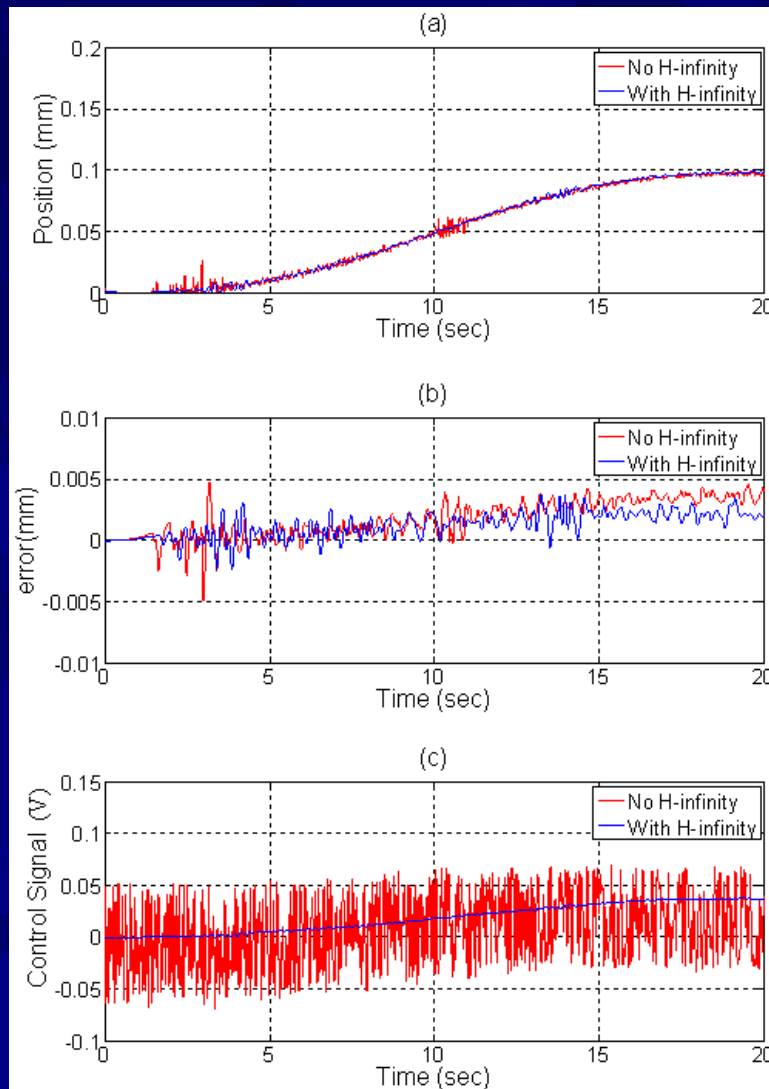
## Case (2)

stroke : 0.1mm

5<sup>th</sup> order : 20sec

Time : 20sec

**System : With  $H_\infty$**



- (a) trajectory response
- (b) error
- (c) control signal



# Experimental comparison of different strokes

Case (1)

stroke : 0.1mm

5<sup>th</sup> order : 20sec

time : 20sec

Case (2)

stroke : 0.15mm

5<sup>th</sup> order : 20sec

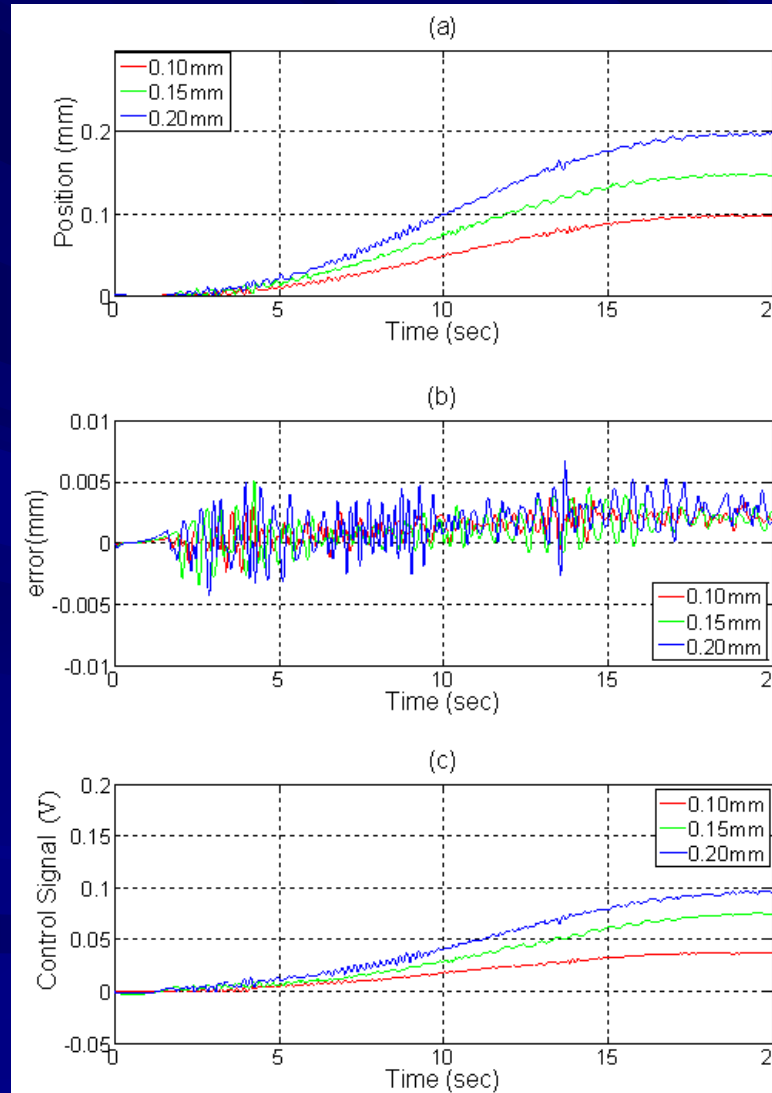
time : 20sec

Case (3)

stroke : 0.2mm

5<sup>th</sup> order : 20sec

time : 20sec



(a) trajectory response  
(b) error  
(c) control signal



# Experimental comparison of different loading

## Case (1)

Stroke: 0.1mm

5<sup>th</sup> order : 20sec

Time : 20sec

**Thickness: 1mm**

## Case (2)

Stroke: 0.1mm

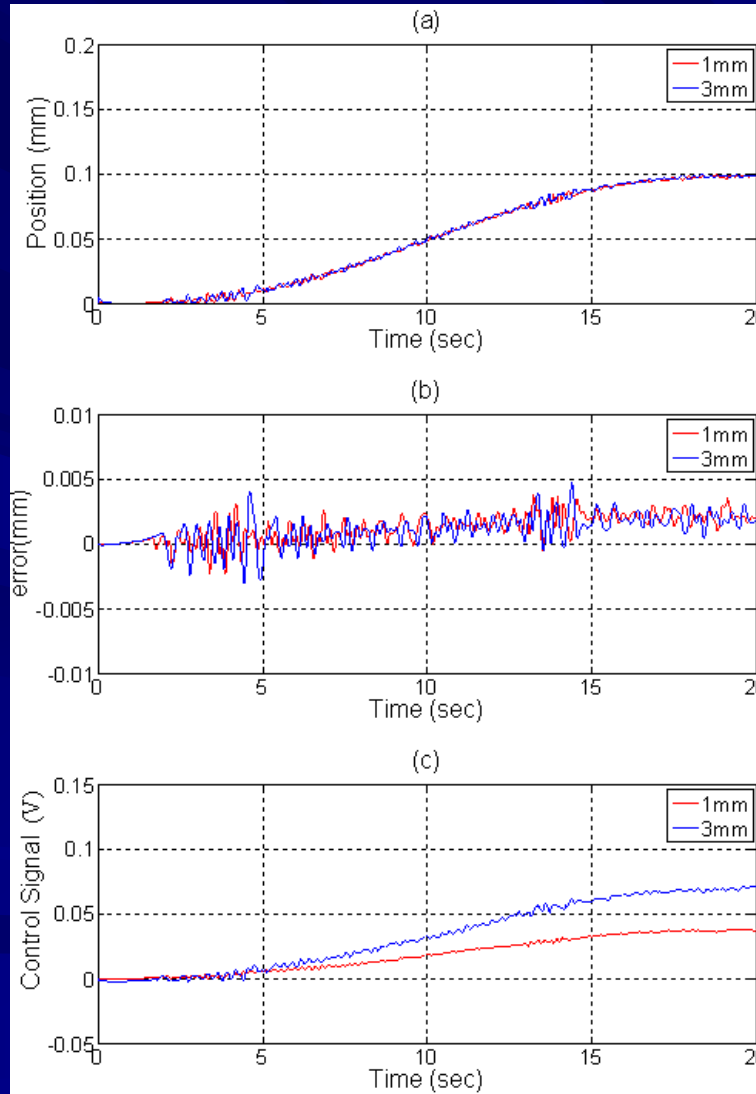
5<sup>th</sup> order : 20sec

Time : 20sec

**Thickness: 3mm**



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**No.46**



- (a) trajectory response
- (b) error
- (c) control signal



# Experimental comparison of different sinusoid tracking control

Case (1)

stroke : 0.05mm

period :  $2\pi$ sec

time : 20sec

Case (2)

stroke : 0.075mm

period :  $2\pi$ sec

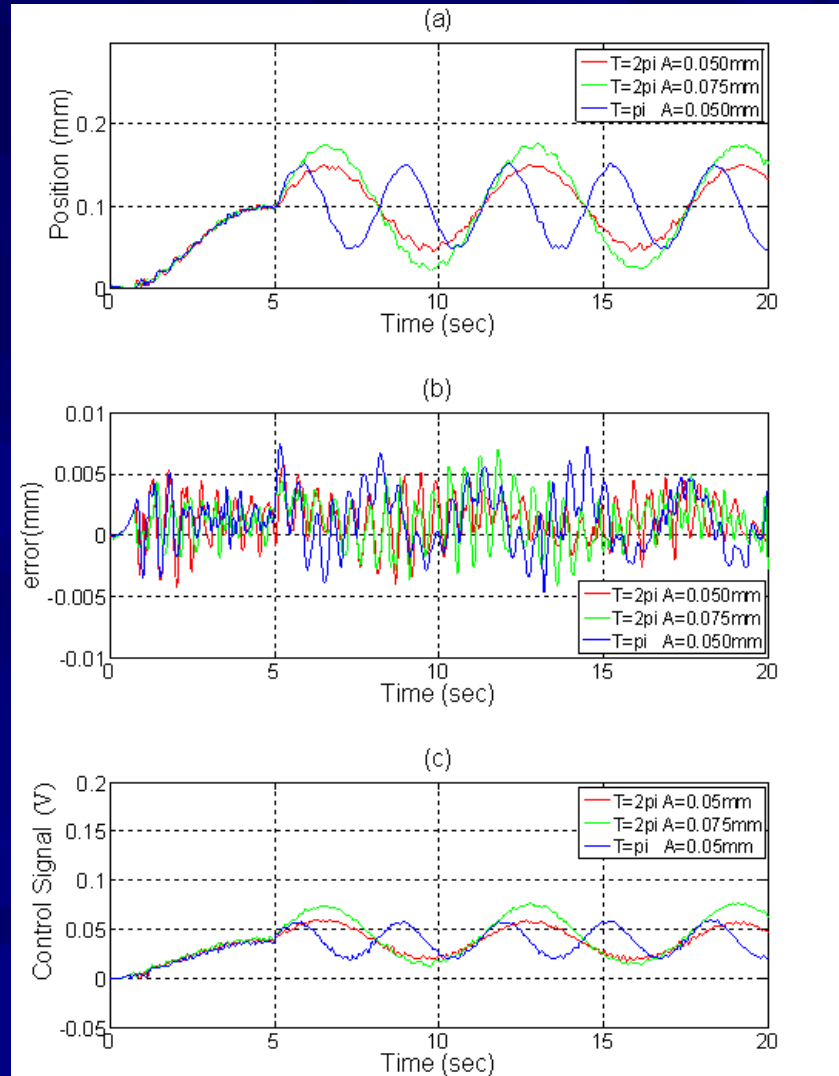
time : 20sec

Case (3)

stroke : 0.05mm

period :  $2\pi$ sec

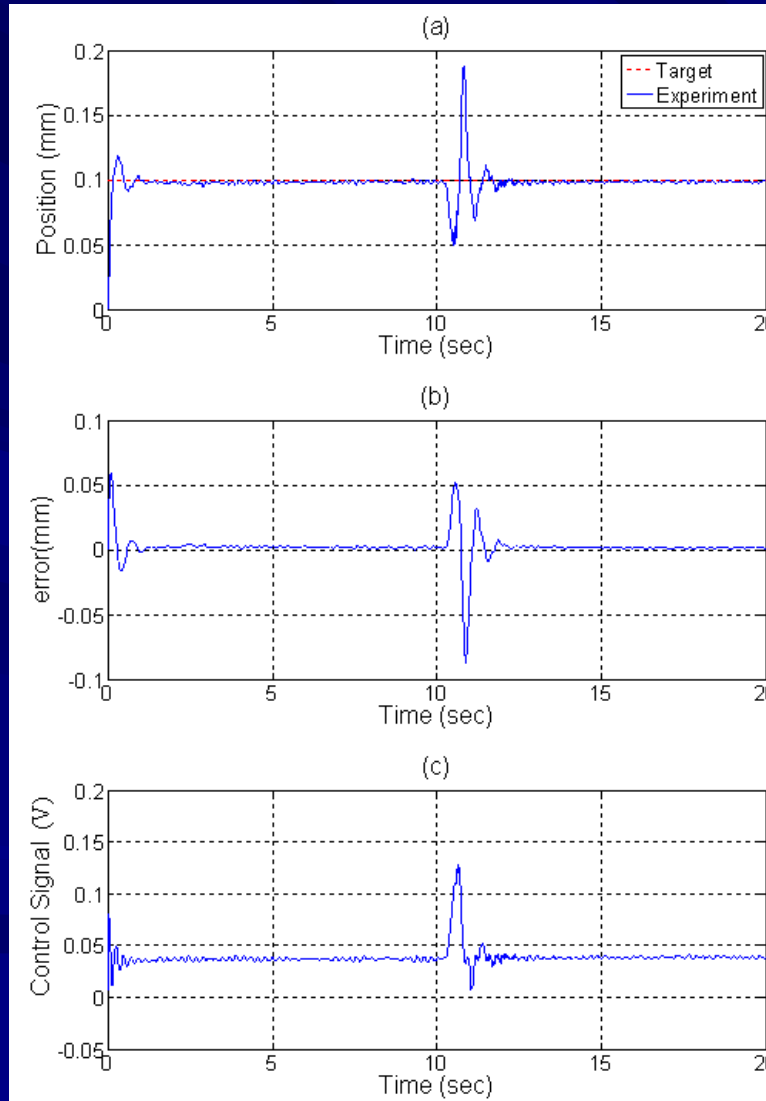
time : 20sec



- (a) trajectory response
- (b) error
- (c) control signal



# Robust test of air bearing conveyor system by adding disturbance (stroke : 0.1mm, 20sec)



- (a) trajectory response
- (b) error
- (c) control signal





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- **Conclusions**



# Conclusions

- This paper developed an **active leveling control** for an air bearing conveyor system for glass substrates of TFT-LCD or solar cell.
- The proposed concept of active leveling control for an **air bearing conveyor system** is to input high pressure air to the air bearing plates such as to form an **air cushion** between the air bearing plates and the glass substrates.



# Conclusions

- Hence, the **friction** between the glass substrates and the air bearing plates **can be avoided**, and the damage of the friction or shaking on the surface of glass substrates can be diminished.
- Through **simulation and experiments**, the unstable leveling position of the glass substrates occurring due to **unstable air pressure** between the air bearing plates and the glass substrates can be **improved** by means of the closed-loop leveling position control.



# *Thanks for Your Attention!*

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